PDHonline Course M335 (8 PDH)

Commercial HVAC

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Overview

Commercial air-conditioning or HVAC (Heating, Ventilation and Air Conditioning) refers to the mechanical systems which control temperature, humidity, air flow, and air quality in homes and commercial buildings. HVAC refrigeration equipment is generally sized in tons or Btu per hour and the heating equipment is specified in kW or Btu per hour. Each refrigeration ton equals to the heat extraction rate of 12,000 Btu per hour.

A HVAC conditioning system includes more than just the air conditioning unit itself. A complete system also includes the air distribution system (ductwork, dampers, grilles and registers), hydronic/refrigerant piping and the temperature and schedule control system. Each of these components makes an important contribution to the performance and efficiency of the system as a whole. In order to operate efficiently, a system needs to be properly sized and installed. Oversized units cost more to operate and do a poor job of comfort control, and poor installation can dramatically reduce the as installed efficiency of the system. The controls are also an integral part of the system and should include programmable thermostats and timers for scheduling of air conditioning equipment or a computerized energy management systems (EMS).

HVAC systems also make up approximately 50% of energy usage in commercial and residential buildings. A relatively small improvement in system design, equipment selection or control strategies can mean greater comfort, lower first costs, easier equipment maintenance, and large long-term savings in energy expenditures over the life cycle of the system. Here are three major factors to keep in mind before investing in a commercial HVAC system:

HVAC Systems: The HVAC components may be assembled into systems literally dozen or hundred different ways, and the choice largely depends on cost, aesthetics and degree of control. Choosing a system with a lower lifecycle cost will have a much bigger impact on your bottom line than choosing the lowest-priced equipment. Taking into account projected expenses such as annual energy usage, installation and maintenance cost, repairs and financing along with the initial cost of the equipment, will help you determine the actual financial impact of your system.

HVAC components - Understand the type of components your HVAC system uses and determine whether it will give you the option to upgrade to more efficient components later. Energy star labeled equipment help reduce energy costs considerably over a standard efficiency system while providing a high level of performance. HVAC rooftop units are now available with efficiency ratings up to 16.1 SEER, 14.3 EER and 16.4 IPLV; systems with a higher IPLV rating can help reduce energy use year-round, especially during spring and fall when only part-load operation is needed. Many regional utilities offer rebates to make high-efficiency equipment more attractive to commercial customers.

Control systems: Investigate advanced control systems to take your comfort control to a higher level. The control systems make it possible to improve energy efficiency, simplify troubleshooting, even monitor and control a wide range of fire and life safety systems, access and security control technology, lighting equipment and maintenance management programs on the same network. Look for a control system that offers open protocol options such as LonTalk® and BACnet® to take advantage of increased integration with other building systems.

This course provides an overview of above criteria in detail and is divided in 3 modules:

Section -1 HVAC Systems

Section -2 HVAC System Components

Section -3 HVAC Control System Equipment and Control Loops

SECTION -1 HVAC SYSTEMS

The most common air-cooling systems are either direct expansion (DX) type or the chilled water type.

DIRECT EXPANSION (DX) SYSTEMS

In direct expansion (DX) systems, the air is cooled with direct exchange of heat with refrigerant passing through the tubes of the finned cooling coil. All these systems comprise of a hermetic sealed or open compressor/s, evaporator (cooling coil fabricated out of copper tubes and aluminum fins), a supply air blower, filter, a condenser and heat rejection propeller fan. These come in two types:

Unitary System – The most common types of DX systems are unitary air conditioners and heat pumps. "Unitary" refers to the fact that all of the components necessary to heat, cool, dehumidify, filter and move air are included in one or more factory-made assemblies. Since all equipment is prepackaged, the installation cost is usually lower, and the performance quality is often higher than field-erected systems. Unitary equipment is available as single package or as split systems.

Single package units include all of the necessary functions and components in one package that is installed outside the building. Window air-conditioners, package units are typical example of unitary DX systems.

- Room air conditioner (capacity range of 0.5 to 3 TR per unit, suitable for an area of not more than 1000 square feet).
- Packaged unit integral air-cooled condenser (capacity range of 3 to 50 TR, suitable for a maximum an area of 1000 – 10000 square feet).

Split System – A split system is a combination of an indoor air handling unit (fan and cooling/heating coils) and an outdoor condensing unit (condenser and compressor). Unitary split equipment includes heat pumps and air conditioners with integral or separate gas or electric heating systems. Heat pumps provide both heating and cooling from the same unit and are the most efficient devices. Air conditioners provide cooling only and must be supplemented with either an internal electric or gas-heating coil or with a totally stand-alone heating system. Split-systems are typically found in residential and small commercial installations with capacity ranges varying 1 to 50 TR and suitable for an area of 100 – 10000 square feet. The new ductless systems which can be conveniently mounted on the ceiling or wall are in this family.

Performance Ratings Terms for Unitary Equipment

- **SEER** – The Seasonal Energy Efficiency Ratio is a representation of the cooling season efficiency of a heat pump or air conditioner in cooler climates. It applies to units of less than 65,000 Btuh capacity. The higher the SEER rating, the more efficient the AC system operates.

- **EER** – The Energy Efficiency Ratio is a measure of a unit's efficiency at full load conditions and 95 degrees outdoor temperatures. It typically applies to larger units over 65,000 Btuh capacity.

- **HSPF** – The Heating Season Performance Factor is a representation of the heating efficiency of a heat pump in cooler climates.

- **Btuh** – Btuh or Btu/h is a rate of heating or cooling expressed in terms of British Thermal Units per Hour.

- **Ton** – One ton of cooling is the energy required to melt one ton of ice in one hour. One ton = 12,000 Btuh

Efficiency Ratings of Unitary Equipment

Federal law mandates a minimum efficiency of 10 SEER for unitary equipment of less than 65,000 Btuh capacity. The American Society of Heating, Refrigeration and Air Conditioning Engineers (ASHRAE) recommend 10 EER for equipment between 65,000 and 135,000 Btuh. ASHRAE standard 90.1 recommends other efficiencies for larger equipment. It is often cost effective to pay for more efficient equipment. For example, upgrading from a 10 SEER to a 12 will reduce cooling costs by about 15 percent. Upgrading from a 10 to a 15 reduces cooling costs by about 30 percent.

CHILLED WATER SYSTEMS

In a chilled water system, liquid water is pumped throughout the building to “chilled water coils”. Since the liquid water needs to be at a cold temperature, a “cooling plant” is required. The plant is typically referred to as a chiller plant. These are usually pre-packaged by the manufacturer with the evaporator and condenser attached, so that only water pipes and controls must be run in the field.

Chilled water systems are further categorized as air-cooled or water cooled system depending on how the heat is rejected out of the system. The chilled water system is also called central air conditioning system. This is because the chilled water system can be networked to have multiple cooling coils distributed through out a large or distributed buildings with the refrigeration equipment (chiller) placed at one base central location. Chilled water systems are typically applied to the large and/or distributed areas. Capacity ranges from 20- 2000 TR and are suitable for an area of 3000 square feet and above.

AIR CONDITIONING SYSTEM DESIGN CONFIGURATIONS

The air-conditioning components and equipments may be designed and assembled in literally dozen or hundred different ways but in practice these are broadly classified into three categories:

1. **Centralized Ducted “All – Air” Systems -** These are systems in which the primary movement of heat around the building is via heated and cooled air. These systems are the most common in large spaces such as office buildings, common public areas, retail, shopping, manufacturing areas, airports, hotel lobbies etc.

2. **Centralized Fluid Based Hydronic Systems -** These are systems in which a fluid - typically water but possibly refrigerant - is used to move heat around the building. These systems are fairly common in office rooms, hotel rooms, schools, building perimeter control etc.

3. **Decentralized Systems -** These are systems in which heating and cooling is conducted locally, with little or no bulk movement of heat around the building. Individual unit ventilators are dispersed in small rooms and around perimeter of a building. These systems are relatively common in schools, small hotels, domestic applications, residential homes and small offices.

The boundaries between these system types are not absolute, but they form useful categories within which to put the many different systems. The choice largely depends on the following -

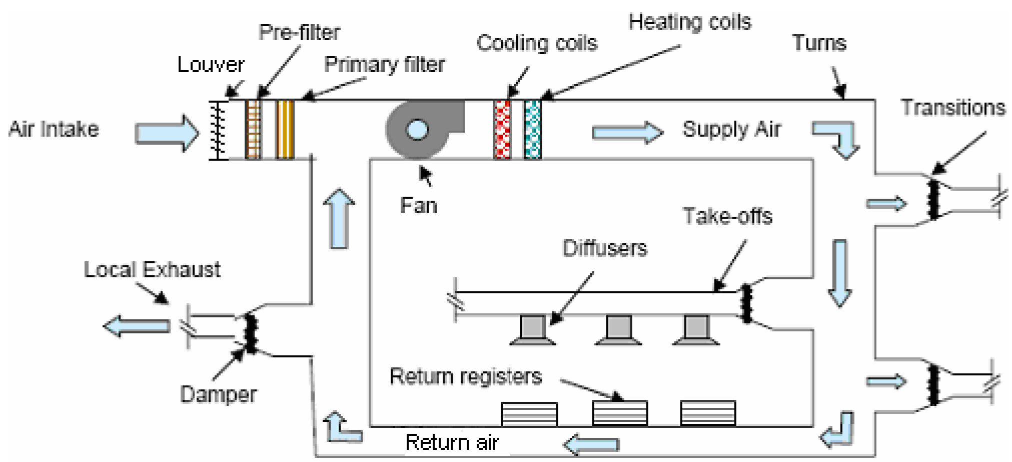
1. System constraints - Cooling load, zoning requirements, acceptable tolerance to temperature/humidity, degree of control etc
2. Architectural Constraints - Size and appearance of terminal devices, acceptable noise level, Space available to house equipment and its location relative to the conditioned space, acceptability of components obtruding into the conditioned space
3. Financial Constraints - Capital cost, Operating cost, Maintenance cost

We will review some of the options and issues under each of these categories.

ALL – AIR SYSTEMS

In an 'All-Air system', the refrigerant or chilled water is used to cool and dehumidify the air in the air handling unit (AHU). The cool air is then circulated throughout the building thru the ductwork. Heating can also be accomplished either by hot water or electrical strip heaters. The centralization of these systems allow for better management and system operation. On the other hand, they also require either a mechanical room adjacent to the controlled space for locating the AHU and large ductwork in building space.

The diagram below indicates the main components of a typical air-conditioning system.



Typical air-conditioning system

This is an air-based system, which is the most dominant air-conditioning type for large buildings. Fresh air is drawn into the building through the intake louver, mixed with return air, heated or cooled to a controlled temperature, circulated around the building and provided to the occupied space. Local temperature control is provided by a terminal reheat unit attached to a temperature controller within the occupied space. Exhaust air is extracted from the space and dumped to the outside. In general, the majority of the return air is recycled via the return air duct. The individual components of this system are:

1. **Air Handling Unit –** This is a cabinet that includes or houses the central furnace, air conditioner, or heat pump and the plenum and blower assembly that forces air through the ductwork.

2. **Intake louvers -** These are the external louvers through which supply air is drawn into the building. Intake is generally equipped with volume control damper to regulate the amount of fresh air and economizing the quantity of outside air during favorable outside conditions.

3. **Filters -** These are used to remove particles of dust or dirt from the supply air.

4. **Heating coils -** These heat up the incoming airstream using coils through which hot water is passed or banks of electric heating elements.

5. **Cooling coils -** These cool the incoming airstream using coils through which refrigerant or water is passed.

6. **Supply fans -** These are used to circulate the air through the network of ductwork.

7. **Ductwork –** It is a branching network of round or rectangular tubes generally constructed of sheet metal, fiberglass board, or a flexible plastic and wire composite material located within the walls, floors, and ceilings. The three most common types of duct material used in home construction are metal, fiberglass duct board, and flex-duct.

8. **Supply Ductwork -** These carry air from the air handler to the rooms in a house. Typically each room has at least one supply duct and larger rooms may have several.

9. **Return Ductwork –** These carry air from the conditioned space back to the air handler. Most houses have only one or two main return ducts located in a central area.

10. **Supply and Return Plenums**- These are boxes made of duct board, metal, drywall or wood that distribute air to individual ducts or registers.

11. **Terminal reheat heating coils-** These use hot water coils or electric heating elements to heat up the air being supplied to one part of the building according to the temperature in that space.

12. **Supply and extract grilles -** These are the points at which the air is either supplied into or extracted from the space, and may be ceiling-mounted or wall-mounted. Also called diffusers or registers.

13. **Boots**- These connect ductwork to registers.

14. **Extract fans -** These are used to extract the air from the space and discharge it to outside.

15. **Return air duct -** These are interconnections between inlet and outlet ductwork sections, which let a controlled amount of air recirculate around the air conditioning system when full fresh air is not required.

16. **Exhaust louvers -** These are the external louvers through which extract air is discharged from the building.

CENTRALIZED AIR CONDITIONING SYSTEMS

System Pros	**System Cons**
1. The central plant is located in unoccupied areas, hence facilitating operating and maintenance, noise control and choice of suitable equipment. 2. No piping, electrical wiring and filters are located inside the conditioned space. 3. Allows greater energy efficiency by using greater amount of outside air during of favorable seasons instead of mechanical refrigeration. 4. Seasonal changeover is simple and readily adaptable to climatic control. 5. Gives a wide choice of zonability, flexibility, and humidity control under all operating conditions. 6. Heat recovery system may be readily incorporated. 7. Allows good design flexibility for optimum air distribution, draft control, and local requirements.	1. Requires additional duct clearance which can reduce the usable floor space. 2. Air-balancing is difficult and requires great care. 3. Accessibility to terminals demands close cooperation between architectural, mechanical and structural engineers.

System Pros	**System Cons**
8. Well suited to applications requiring unusual exhaust makeup. 9. Infringes least on perimeter floor space. 10. Adapts to winter humidification. 11. Are more energy efficient than decentralized systems.	

TYPES OF "ALL-AIR" SYSTEMS

There are three major types of Centralized Ducted Air Systems:

1. Constant Volume Systems
2. Dual Duct Systems
3. Variable Volume Systems

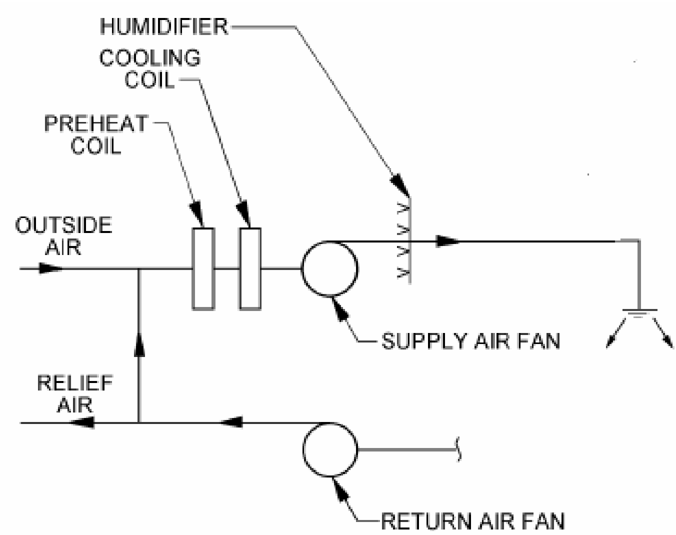
CONSTANT VOLUME SYSTEMS (CAV)

Constant volume systems deliver a constant volume of air and responds to changing thermal loads by varying air temperature. The air volume is usually based on the design cooling load for the given zone. The ducting and air handling system are sized to match the heat gain from equipment, lights, exterior, and people. Typical applications include:-

1. Space with uniform loads (small office buildings, manufacturing plants, retail etc.)
2. Small spaces requiring precision control

System Description

Constant volume systems are common form of air-conditioning of single thermal zones and are often the system of choice due to simplicity, low cost and reliability. Air is drawn from outside, filtered and then heated or cooled as required. A supply fan then distributes the air through a ductwork network to supply grilles in the space. Air is drawn from the space via extract grilles. The air is then recycled via the return air duct or discharged to the outside through external discharge louvers.



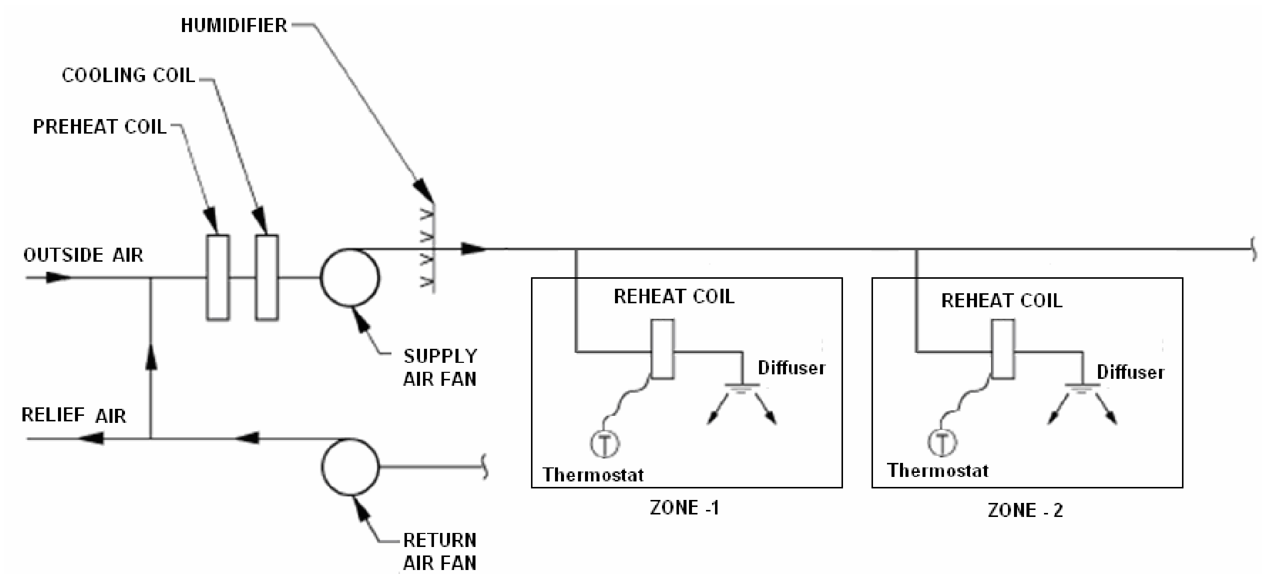
Constant Volume System

Normally, the equipment is located outside the conditioned space but can also be installed within the conditioned are if conditions permit.

Controls Configurations

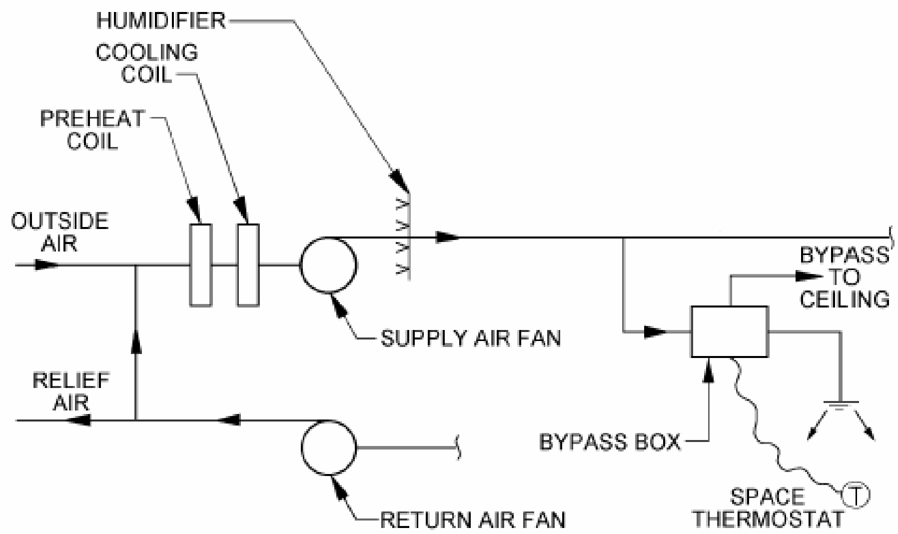
Most CAV systems are small, and serve a single thermal zone. One air handling system is required for each zone because there can be only one supply air temperature at any given time. Air temperature can be varied in the air handling unit to meet the sensible heating or cooling requirement of the zone.

For the typical system serving multiple zones, not all zones will need full cooling at the same time due to unequal loading. For example the perimeter areas with different solar exposures will see different loads through out the day, while the interior zones will see a fairly constant load. Because the system cannot vary the supply air volume to each zone, some zone air handling systems will have to supply warmer air to maintain the space setpoint. This is achieved by terminal reheat. The terminal reheat units are controlled by a thermostat in the occupied space and apply heating to adjust the air supply temperature to that required to maintain comfortable conditions in the room. This arrangement provides for simple control but is very inefficient, as the supply air is often both cooled and heated, resulting in energy waste.



Constant Volume System with Reheat

Alternative control arrangement for CAV system serving multiple zones is the use of “Bypass box”. In this arrangement a bypass box with motorized relief damper is added to the branch duct serving a zone. In response to the zone thermostat signal, the relief damper discharge part of cool air to the open return ceiling thereby restricting the amount of cold air supply to the zone. This is more energy efficient way compared to reheat arrangement but is typically applied on open return above false ceiling.



Constant Volume System with Bypass Box

Effectiveness and Internal Comfort

Constant volume systems are the simplest to install, but all areas receive a preset percentage of the cooling all the time. However, loads can vary dramatically: the number of people, appliances and lights, and solar gain through the windows are unpredictable. With this system, only the area near the thermostat is satisfied all the time. Everyone else is either too hot or too cold.

One path around this problem is to locate the main thermostat in the area with the highest cooling load. Areas requiring less cooling can then have their air flows reheated. This however is a highly energy inefficient method and is prohibited by many codes. One energy efficient system for retrofit or new construction is the variable air volume (VAV). In this system, the quantity of air is reduced to zones with reduced cooling requirements. We will discuss this further in following paragraphs.

As a generic rule, for the office areas, large interior zones can be provided with constant volume systems (apply VAV system for perimeter zones). Similarly for shopping corridors, hotel lobbies, airport concourse can have constant volume systems whereas for individual shops, hotel rooms etc should have individualized VAV controls.

Energy Saving Considerations

Constant volume, variable temperature systems use more fan work than VAV systems because they are usually designed for a larger design supply air flow rate, and they cannot modulate the supply air flow rate during part load conditions. The following are some considerations outlined in ASHRAE STD 90.1-1999. The numbers in brackets refer to Std. 90.1-1999 sections.

1. Equipment must be scheduled off automatically during unoccupied hours [6.2.3.1].

2. Adjustment of economy cycle operation to ensure maximum gains are made from the use of fresh air for "free" cooling. Demand Controlled Ventilation is required for systems with at least 3,000 CFM of outdoor air and occupant density greater than 100 people per 1,000 ft^2 [6.2.3.9].

3. Air- or water-side economizers are required. There are several exceptions to this rule, particularly when dealing with heat recovery [6.3.1].

4. Where humidification is required to maintain humidity above 35°F dewpoint, water-side economizers must be used when economizers are required. Introducing large amounts of cool, dry air while meeting the sensible cooling load adds significantly to the humidifier load. Process loads, including hospitals, are exempt [6.3.2.4].

5. Energy recovery is required for systems with at least 5,000 CFM supply air and a minimum of 70% outdoor air. This is specifically aimed at schools and labs [6.3.6.1].

6. For systems under 20,000 CFM, constant volume fans are limited to 1.2 hp/1,000 CFM. For systems over 20,000 CFM, fans are limited to 1.1 hp/1,000 CFM [6.3.3.1].

CONSTANT VOLUME SYSTEMS

System Pros	**System Cons**
1. Easy to design and install 2. Dedicated unit per zone offers good	1. Supply air volume cannot be varied. 2. Ducting is oversized and there is a

System Pros	**System Cons**
temperature control and redundancy. 3. Air- or water-side economizers can be added easily to the design to minimize mechanical cooling during cooler weather. 4. The main air handling systems can accommodate the ventilation air, avoiding dedicated ventilation equipment. 5. Constant volume systems have relatively few moving parts and thus can operate with a reasonable level of reliability.	significant fan power penalty. 3. Varying the supply air temperature does not guarantee dehumidification. 4. Many small systems are needed, each requiring access to the zone it serves. 5. Rooftop and self-contained systems offer limited cooling diversity among different zones. 6. The control of terminal reheat units is another source of problems. The operation of the thermostat and the valve controlling the terminal reheat unit (where a hot water coil is used) needs regular checking and calibration to ensure that good quality control is maintained.

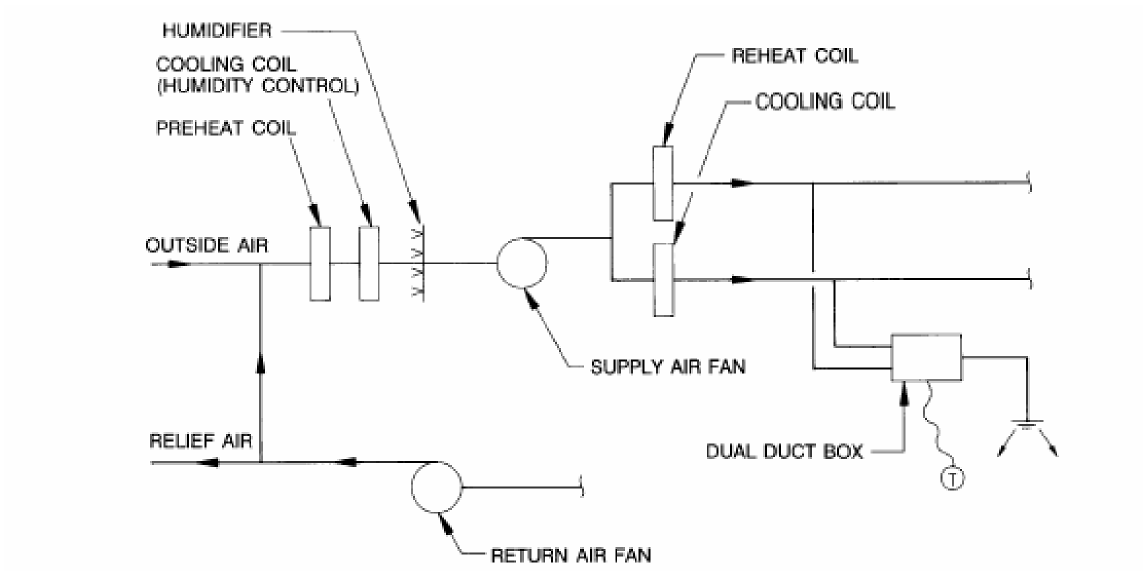
DUAL DUCT SYSTEM

Dual or double duct system is an air conditioning system in which cold and warm air is circulated throughout a building via two parallel ducts. Hot air flows within one duct, cold air within the other. The proportion of hot and cold air delivered to each room within the building may be controlled by thermostatically operated dampers on the ducts outlet. The system is well suited to provide temperature control for individual spaces or zones. Common applications include:

• Office Buildings

• Institutional

System Description



Dual Duct Constant Air Volume System

Dual-duct all-air systems use two ducts, the first supplying warm air and the second supplying cold air. Air is mixed at the terminal serving the individual space, so that supply air of the desired temperature is delivered to the space. Dual-duct systems are uncommon in new buildings.

Controls Configurations

There are a number of ways in which a dual-duct system can be controlled. For a typical system serving multiple zones, the air is divided into two, with one supply duct being heated (the "hot deck") and the other cooled (the "cold deck"). In the simplest form of control, the hot deck operates at maximum supply temperature, which is around 84-86°F, while the cold deck operates at minimum supply temperature, which is around 50-55°F. A thermostat in the occupied space controls a mixing box which adjusts the balance of hot and cold air being supplied, to maintain comfortable conditions. The ultimate in control for a system of this type is to adjust the hot and cold deck temperatures on a continuous basis to match the heating and cooling needs of the building.

Effectiveness and Internal Comfort

Dual-duct systems can provide a good quality of servicing but are expensive to install. The quality of temperature control is mainly limited by the way in which rooms are grouped together for temperature control. Often multiple rooms or even the entire face of a building may be served as a single space with only one point of temperature control. This can cause discomfort if individual rooms within this space have different amounts of equipment or sunlight.

DUAL DUCT SYSTEMS

System Pros	**System Cons**
1. Only the required amount of supply air is used, so fan power is not wasted. 2. Systems with terminal volume regulation are self-balancing. 3. Zoning of central equipment is not required. 4. Instant temperature response is achieved because of simultaneous availability of cold and warm air at each terminal unit. 5. No seasonal changeover is necessary. 6. There is no simultaneous heating and cooling. Heat in plenum air can be	1. Initial cost is usually higher than other VAV systems. Two ducting systems and two air handling units add to cost. 2. Does not operate as economically as other VAV systems. 3. Introducing the correct volume of outdoor air into the building is more difficult. 4. Providing each zone with the correct amount of outdoor air is more difficult. 5. More sophisticated controls are required. 6. Large duct shafts from centralized air handling systems are required. 7. Dual-duct systems have critical moving

System Pros	**System Cons**
used as reheat. 7. Fixed cold air supply temperature maintains humidity control in space. 8. Air- or water-side economizers can be added easily to the design to avoid mechanical cooling during cooler weather. 9. The main air handling systems can accommodate the ventilation air, avoiding dedicated ventilation equipment.	parts in the form of the mixing boxes. The operation of the mixing boxes requires regular maintenance to ensure that the overall performance of the system is maintained. As with other air-based systems, the balancing of air-flows around the system is critical to the system operation. Regular rebalancing of air-flows is highly recommended to maintain performance.

VARIABLE AIR VOLUME SYSTEMS (VAV)

Variable air volume system (VAV) delivers a constant temperature of air and responds to changing thermal loads by varying the quantity of supply air.

Conditioned spaces in buildings are primarily divided into two categories: (1) Perimeter zone, where there are external wall, roofs, and windows and space load varies depending on solar heat, outdoor-indoor temperature difference and internal load; and (2) Interior zone where space loads are mainly internal loads. In an interior zone, the air system often operates at cooling mode in both summer and winter. To account for load variations (in perimeter and interior zones), VAV system varies the air volume in response to the varying load to maintain predetermine space parameter and conserve fan power.

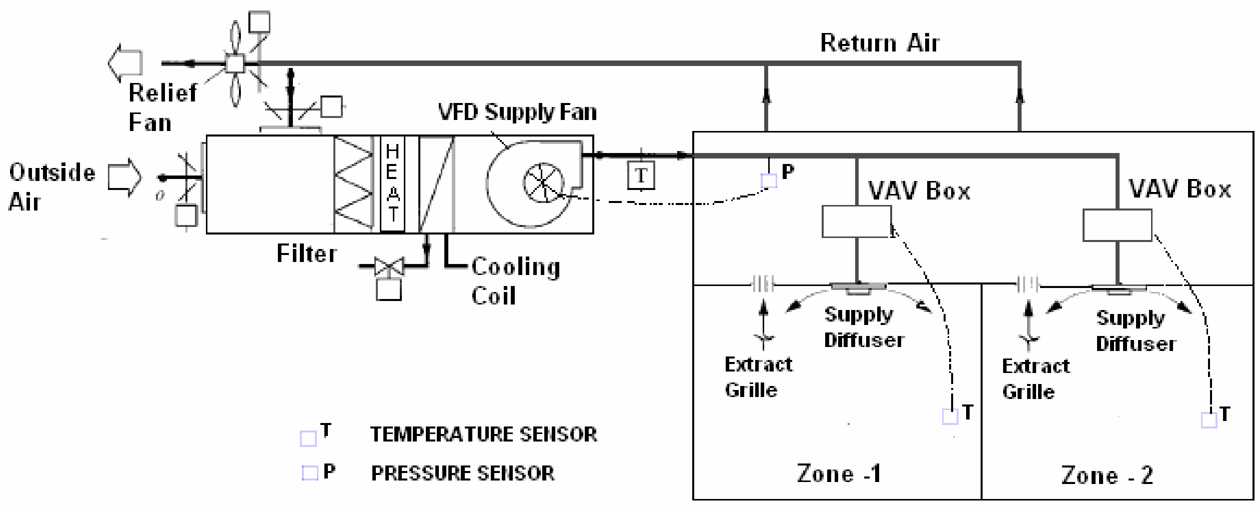
VAV systems are very common for a wide range of applications. Common applications include:

1. Office Buildings

2. Schools

System description

The simplest VAV system incorporates one supply duct which distributes approximately 55ºF supply air. Each branch duct to individual zones having varied load requirements are equipped with VAV unit. Because the supply air temperature is constant, the air flow rate is varied to meet the rising and falling heat gains or losses within the thermal zone served.



Variable Air Volume System

Variable air volume system with terminal reheat

Variable air volume (VAV) systems use variable volume terminal units or boxes to vary the airflow in each zone or space in accordance to the thermostat signals within the space. Heating is turned on when the air flow reaches a predetermined minimum. If more cooling is required, more cold air is introduced into the space.

In the best implementation of these systems, the central air handling unit fan speed is controlled to maintain a constant duct pressure. An interlock is arranged between the supply and extract fans. Variable air volume systems are very common in larger office buildings.

Controls configurations

Variable air volume (VAV) with reheat systems provide conditioned air to each zone at a constant temperature, typically 55°F. The amount of air varies to match the heat gain from equipment, lights, exterior and people. At part load conditions, VAV systems supply only the necessary amount of conditioned air to each zone, saving significant fan energy.

A zone temperature sensor is used to control a damper that is arranged to maximize air-flow when heating or cooling loads are high and cut down to minimum air when loads are low. When more dampers close, the duct system static pressure increases. The primary supply fan adjusts to maintain duct static pressure using discharge dampers (FC fans only), inlet guide vanes, or variable frequency drives (VFDs). Control of the system's fan capacity is critical in VAV systems. Without proper and rapid flow rate control, the system's ductwork, or its sealing, can easily be damaged by over-pressurization. Automatic controls for the fan generally comprise of a speed modulating control to the fan motor to maintain a constant pressure in the supply duct. However, in some systems, variable volume control may be achieved by other methods, such as a damper restricting the flow or the adjustment of fan blades.

The control of supply duct temperature is similar to that of a constant volume system. In many buildings, the air in the supply duct is heated or cooled to a constant temperature of typically 14-16oC. The terminal reheat units are controlled by a thermostat in the occupied space and apply heating to adjust the air supply temperature to that required to maintain comfortable conditions in the room. Because the airflow is variable, this simple control arrangement does not produce the inefficiencies present in constant volume systems. More complex control of duct temperature is feasible, following the same "hi select" methods as described for constant volume systems. However, as long as the minimum air-flow in the system is substantially smaller than the maximum airflow, and the variable volume components of the system are operating correctly, additional control complexity is not necessary.

Effectiveness and internal comfort

VAV systems provide very good control and a high level of flexibility in operation. The internal comfort control is generally good, but there can be problems when air flows are low.

In particular, there may be problems with ensuring that adequate fresh air is supplied to individual zones and also low flows of air mix poorly with the air in the room and instead drop directly from the supply diffuser, causing discomfort for anyone sitting immediately below. There can also be problems with air "whistling" past the variable volume dampers under certain conditions. However this can be offset by setting the VAV boxes to provide a minimum amount of air, even if that amount exceeds cooling requirements. While this maintains air turnovers and minimum outdoor air ventilation rates, the zone may be overcooled and reheat may be required, resulting in simultaneous heating and cooling. ASHRAE Standard 90.1-1999 requires the minimum supply air volume be the ventilation rate for the zone for most applications.

A properly functioning variable air-volume system is very efficient because it has minimal simultaneous heating and cooling and low fan energy.

Type of Variable air volume terminal units

There are generally five different types of units that are used: pressure-independent volume units; pressure-dependent, airflow-limiting, maximum volume units; pressure-dependent units; bypass (dumping) units; and supply outlet throttling units. One other type of unit that is used is the fan-powered variable air volume terminal unit.

1. **Pressure – Independent Units -** Pressure-independent volume units regulate the flow rate in response to its respective thermostat's call for heating or cooling. The thermostat controls airflow to the space by varying the position of a simple damper or volume regulating device located in the unit. The required flow rate is maintained, regardless of the fluctuation of the system pressure being supplied by the air handling unit supply fan. These units can be field- or factory-adjusted for maximum and minimum airflow settings.

2. **Pressure Dependent-Airflow Limiting Maximum Volume Units** - A pressure-dependent, airflow-limiting, maximum volume unit regulates maximum volume, but the flow rate will oscillate when system pressure varies. These units are less expensive than pressure-independent units. These units can be used where pressure independence is required only at maximum airflow, where system pressure variations are relatively small, and where some degree of fluctuation or "hunting" is tolerable.

3. **Pressure Dependent Units -** Pressure-dependent units do not regulate the flow rate, but position the volume regulating device in response to the thermostat. These units are the least expensive and should only be used where there is no need for maximum or minimum airflow control and the air handling unit system pressure is stable.

4. **Bypass Dumping Units** - Generally, in small air handling systems, the cost of a variable air volume system is too high. However, by using bypass (dumping) units in certain zones or spaces, the constant volume system can have variable airflow control. The thermostat controls airflow to the space by varying the position of the volume regulating device. If less air is required to the space, the regulating device closes down and bypasses or diverts some of the air to the return ceiling plenum or return air duct. This technique is very effective for constant volume systems.

5. **Supply Outlet Throttling Units** - Supply outlet throttling units are usually linear diffusers. The area of the throat or the discharge opening varies in approximate proportion to the air volume to maintain throw patterns. The thermostats are usually located at the outlet of the diffuser for easy temperature adjustment. Since these units are pressure-dependent, constant pressure regulators are usually required in the duct system. Noise is a concern when using these units in occupied spaces.

6. **Fan Powered Variable Air Volume Units** - Fan-powered variable air volume units are available in two types: parallel and series flow units. The units have the same components as pressure-dependent or pressure-independent volume units, and in addition, a fan and usually an electric or hot water heating coil. Fan-powered variable air volume units, both series and parallel, are often used for building perimeter heating, because they move more air through a room at low cooling loads and during the heating cycles compared to variable air volume reheat or perimeter radiation systems.

Energy Considerations

Varying the supply air volume reduces fan work, a major use of building energy. The following are some considerations outlined in ASHRAE Std 90.1-1999. The numbers in brackets refer to Std. 90.1-1999 sections.

1. Equipment must be scheduled off automatically during unoccupied hours [6.2.3.2].

2. Demand Controlled Ventilation is needed for systems with at least 3,000 CFM of outdoor air and an occupant density greater than 100 people per 1,000 ft^2 [6.2.3.9].

3. Where humidification is needed to maintain humidity above 35°F dewpoint, water-side economizers must be used when economizers are needed. Introducing large amounts of cool, dry air while meeting the sensible cooling load adds significantly to the humidifier load. Process loads, including hospitals, are exempt (6.3.2.4). • For systems under 20,000 CFM, VAV is limited to 1.7 hp/1,000 CFM. For systems over 20,000 CFM, VAV systems are limited to 1.5 hp/1,000 CFM [6.3.3.1].

4. 30 hp and larger fan motors must use no more than 30% of design power at 50% airflow [6.3.3.2].

5. Energy recovery is required for systems with at least 5,000 CFM supply air and a minimum of 70% outdoor air. This is specifically aimed at schools and labs [6.3.6.1].

VARIABLE AIR VOLUME SYSTEMS

System Pros	**System Cons**
1. Only the necessary amount of primary air is used, conserving primary fan power. 2. Diversity is applied to supply air volume, reducing duct and fan sizes. 3. Capital cost is lower since diversities of loads from lights; occupancy, solar and equipment of as much as 30% are permitted. 4. Virtually self-balancing. 5. It is easy and inexpensive to subdivide into new zones and to handle	1. Difficult to consistently maintain minimum outdoor air quantities entering the building. 2. Difficult to consistently maintain the correct amount of outdoor air in each zone. 3. Requires sophisticated controls. 4. Simultaneous heating and cooling occurs once the minimum air volume is reached in a zone. 5. A separate, distributed heating system is

System Pros	**System Cons**
increased loads with new tenancy or usage if load does not exceed the original design simultaneous peak. 6. No zoning is required in central equipment. 7. Lower operating cost because fans run long hours at reduced volume 8. Refrigeration, heating and pumping matches diversity of loads 9. Unoccupied areas may be fully cut-off 10. Allows simultaneous heating and cooling without seasonal changeover 11. Air- or water-side economizers can be added easily to the design to minimize mechanical cooling during cooler weather. 12. Air handling unit can maintain minimum outside air amounts, avoiding the need for dedicated ventilation equipment.	needed for cooler climates. 6. The variable volume dampers, either as VAV boxes in the ceiling space or as variable volume diffusers, can be maintenance-intensive items. A variable volume system with poorly functioning variable volume functions is in essence just a constant volume system. As a result, regular maintenance and calibration of variable volume equipment is essential for good system performance. 7. If the air supply volume to different spaces falls out of adjustment, then individual rooms and sometimes even entire floors can become quite uncomfortable and the system can become inefficient. 8. The control of terminal reheat units is another source of potential hazards. The operation of thermostats and the valve controlling the terminal reheat unit needs regular checking and calibration to ensure that good quality control is maintained.

CENTRAL FLUID BASED HYDRONIC SYSTEMS

Central fluid based hydronic systems use terminal units to condition the air. The term hydronic refers to an air-conditioning system that uses water for distribution of heat. Cooling and dehumidification is provided by circulating chilled water through a finned coil in the unit whereas heating is provided by supplying hot water through the same or a separate coil.

Why Fluid based Systems?

1. Water has greater specific heat and density compared to air and therefore water can convey a lot of heat per unit volume. The distribution elements (pipes) are typically smaller than that of air distribution ductwork.

2. Zone control is easy and the individual equipment can be shut off during not occupied hours. Typical application is the hotel rooms and multiple small zones.

3. Fluid based systems require small primary air for ventilation purposes. The system relies on 100% recirculation air and there is no return air system.

4. The power required to pump water through the building is usually less than the fan power needed for supply air and return air systems.

5. The fluid based system in combination with provision for ventilation air can match versatility of all-air systems. Positive ventilation, central dehumidification, winter humidification, and good temperature control over a number of control zones are capable with the system.

Centralized fluid-based systems come in a variety of radically different forms. Key types of fluid-based system are:

1. Hydronic system (All – Water System)

2. Fan-coil system

3. Induction system

4. Variable Refrigerant Volume system (VRV)

5. Hybrid system (using both all-fluid and all-water arrangement)

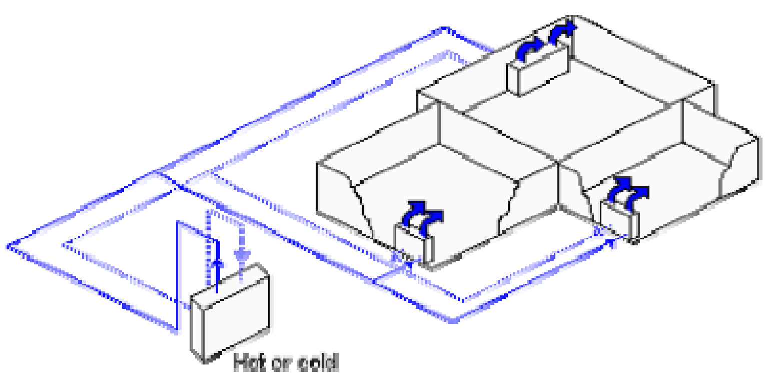
'ALL – WATER' HYDRONIC SYSTEMS

In an all-water hydronic system, the hot or cold-water is distributed to individual heat transfer devices (terminal units) located in each room of the building. When heating, the terminal units draw heat from the water and when cooling these reject heat to the water. The biggest drawback of all-water hydronic system is the difficulty in providing adequate indoor air quality. Due to this concern, these systems should not be applied to the occupied areas and are ideally suited for machinery rooms, substations etc.

An All-Water system uses the following basic components:

1. The use of a chiller (on roofs or plant rooms) to cool the water which would be circulated via circulating pumps to the terminal units (example fan coil units) located in the occupied space.

2. The use of boilers (in plant rooms) to heat the water which would be circulated via circulating pumps to the terminal units located in the occupied space. Hydronic heating systems employ a variety of terminal units that include fan coil units, baseboard radiators, convectors, unit heaters, and radiant floors.

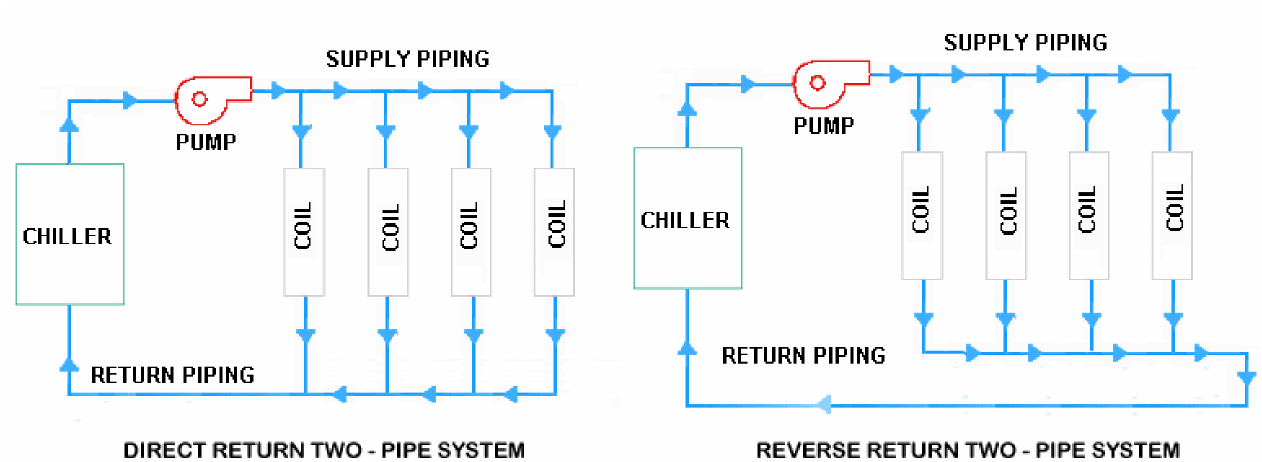


Hydronic systems are designated by how the seasonal switch over from heating to cooling occurs. The hydronic system piping can be arranged in 3 basic configurations: (1) Two-pipe system, (2) Three-pipe system and (3) Four-pipe system.

Two Pipe Systems

Two pipe systems consist of a network of insulated pipes; one pipe supplies chilled water and the second pipe returns it to the chiller. The secondary water is cold in summer and intermediate seasons and warm in winter. With water-changeover, chilled water is circulated during the cooling season and hot water during the heating season. The problems occur during the mid-seasons where cooling can be required part of the time and heating part of the time and no heating or cooling the rest of the time.

Two pipe systems without water-changeover circulate chilled water only and provide heat where it is needed by electric strip heat at the terminal units. In some cases, hot water is circulated during the coldest part of the heating season to reduce operating cost.



The primary air quantity is fixed and the primary air temperature control is achieved by varying the water supply through the coil. When thermostat sensor demand more cooling, the two or 3-way valve located on the line is full open position.

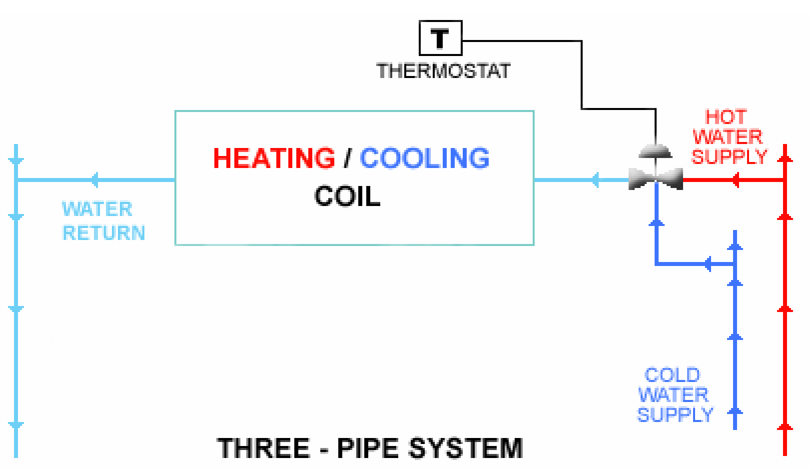
TWO-PIPE SYSTEM

System Pros	**System Cons**
1. Usually less expensive to install than four pipe systems	1. Less capable of handling widely varying loads or providing widely varying choice of room temperature than four-pipe systems

System Pros	**System Cons**
	2. Cumbersome to change over and the problem is acute on the shoulder months (spring and fall) when the occupants need heat in the cool mornings and cooling in the warmer afternoons. **3.** Though it is economic initially on first cost, it is more costly to operate than four-pipe systems.

Three- Pipe System

In the three pipe system, hot water and chilled water are fed to each fan coil, with a common return. This is somewhat more expensive than 2-pipe system, since a third pipe must be run to each unit. Since the hot water and cold water are mixed in the return, these inefficient systems are seldom installed today.



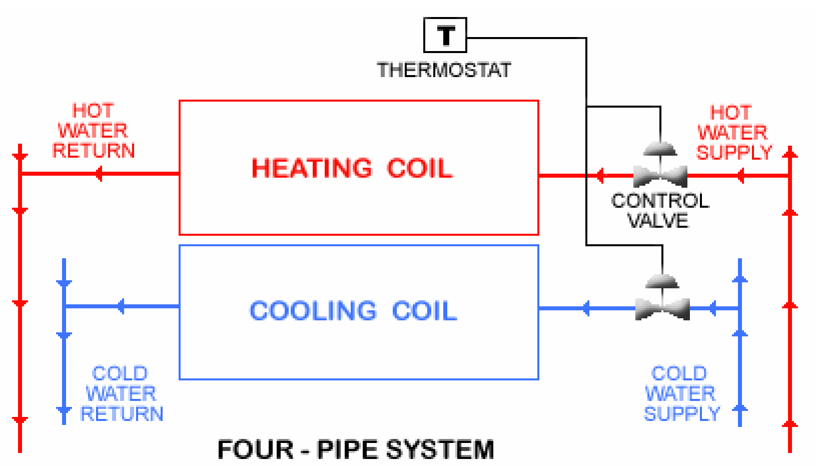
Four-Pipe System

In the four pipe system, the chilled water loop and the hot water loop are completely separate. While these systems are most expensive to install, they are easier to operate in unpredictable climates.

The system is further categories as the independent load system and common load systems.

1) Independent load systems have two separate water coils, one served by hot water, the other by cold water. The systems make use of 2-way on-off valve.

2) Common load systems can have a single coil in the air handler but still supplied independently with 4-pipe system. The systems make use of 3-way diverting valves.



Note that the water flow rate required for heating is much lower than the chilled water flow. It is better to use 4-pipe system, which allows the use of smaller size piping and pumps for lower hot water flow rates.

FOUR-PIPE SYSTEM

System Pros	**System Cons**
1. All year availability of heating and cooling with individual zone temperature control 2. Chilled and hot water could be simultaneously supplied during the Spring and Fall seasons 3. Elimination of zoning cost and complexity 4. Simpler changeover decisions (No summer-winter changeover and primary air reheat schedule) 5. The lowest and quietest fan speed is adequate most of the time. 6. More flexible and adaptable to widely varying loads	1. Four-pipe systems have a high first cost due to the second water system and the need for either two coils or more costly control valves at each terminal unit. 2. The systems also have a high operating cost due to the two pumps operating (though provide good flexibility in meeting varying loads)

HYBRID SYSTEMS

Hybrid systems use both air and water (cooled or heated in central plant room) distribution to room terminals to perform cooling or heating function. Unlike all-water system, this system ensures ventilation air in the spaces so that indoor air quality is not sacrificed.

Air Portion of the System

The air side is comprised of central air conditioning equipment, a duct distribution system, and a room terminal. The supply air, called primary air, usually has a constant volume which is determined by the need for outside (fresh) air for ventilation. When in cooling mode the primary air is dehumidified, to provide comfort and prevent condensation, by a central conditioning unit. In the winter, heating mode, the air is humidified, by the central conditioning unit, to limit dryness.

Water Portion of the system

The water side consists of a pump and piping to convey water to heat transfer surfaces within each conditioned space. The water used can be chilled by direct refrigeration, by using chilled water from a primary cooling system, or by heat transfer through a water-to-water exchanger. Chillers usually supply chilled water anywhere from 35-48°F. Individual room temperature control is by regulation of either the water flow through it or the air flow over it. In the winter, the heating capacity of the coil in a conditioned space must be great enough to heat the space and offset the cool primary air, which is provided.

HYBRID SYSTEMS

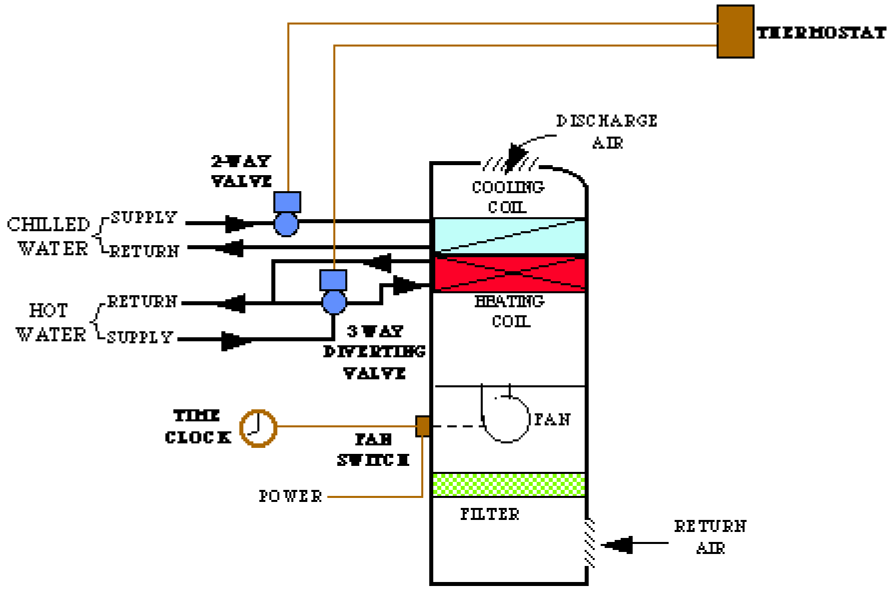
System Pros	**System Cons**
1. Flexible and readily adaptable to many building module requirements. 2. Provides individual room control. 3. The power required to pump water through the building is usually less than the fan power needed for supply air and return air systems. 4. Very good control is available over many zones. 5. Heating and Cooling can be available for all zones allowing variation of loads.	1. No positive ventilation is provided unless wall openings are used. 2. No humidification is provided. 3. Seasonal change over is required. 4. Maintenance and service work has to be done in the occupied areas. 5. Design of between season operations is crucial as a result of the low primary air delivered. 6. Terminal unit filters must be changed often due to secondary airflow.

System Pros	**System Cons**
6. Space needed for distribution system is minimal. 7. Energy saving are achieved by using water instead of air to deliver heating and cooling to the space. 8. Cross contamination is reduced because recirculation occurs within rooms.	

FAN COIL UNITS

A fan coil conditioner unit is a type of room terminal unit that can be used with water-only (all-water) or air-water systems. In the air-water system the fan-coil conditioner unit provides the heating or cooling to a space while the primary air system supplies all ventilation.

System description



Fan-Coil Unit using 4-way piping for heating and cooling along with 2-way and 3-way diverter valve for temperature control

A fan-coil unit is relatively simple air-conditioning equipment that is placed at each place which needs to be heated or cooled. The fan-coil conditioner consists of a fan, filter and coil. The fan draws in room air and passes it through the filter and then by the coil to either heat or cool it and then returns it to the room to control the temperature and particulates. The coils have chilled water or low temperature hot water (LTHW) passing through them, which is supplied at a central location. Where there is a single coil this can be arranged to provide heating and/or cooling according to requirements. Where there are two coils, one is arranged to provide heating and the other cooling. This configuration is referred to as a four-pipe fan-coil unit (as shown in schematic above).

The fan units are available in various configurations that can be either wall mounted or ceiling mounted. Variations may take the form of two-pipe fan-coil units, with a single cooling or heating coil, or may use an electrical heating coil in place of a LTHW coil. Auxiliary air may be delivered to the conditioned space for dehumidification and ventilation purposes.

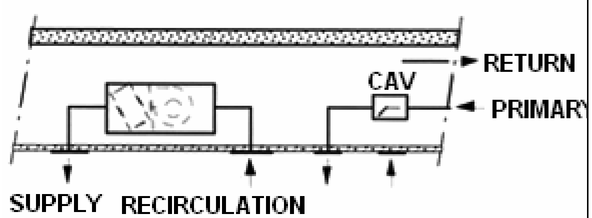
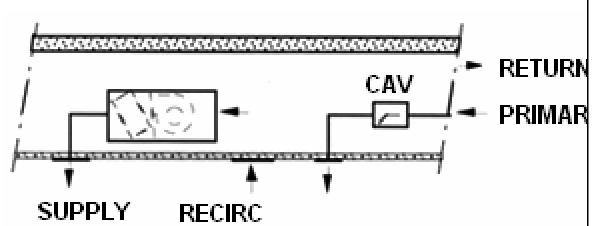
Units are available in standard capacities of 200, 300, 400, 600, 800, and 1200 CFM.

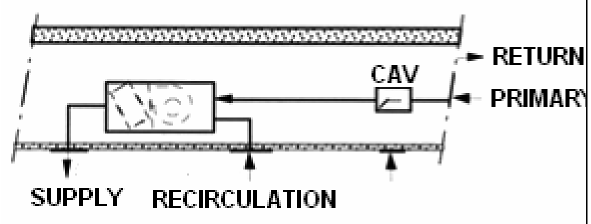
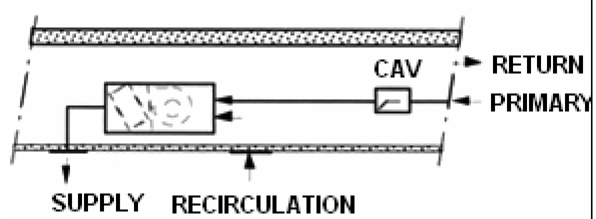
<u>Design Recommendations</u>

In order to make a proper fan coil unit selection, the whole air conditioning system must be looked at because sizing a fan coil unit is not just matching the room load. Other criteria which must be considered during selection are; the minimum room load, fresh air requirements, with or without reheat, 2 or 4-pipe system, zone size, flexibility and maximum noise level in the room, etc.

The cooling capacity which is supplied to the room is the sum of cooling energy available in the primary air and the cooling capacity of the fan coil unit. The way the primary air is supplied to the room determines a great part of the required fan coil cooling capacity.

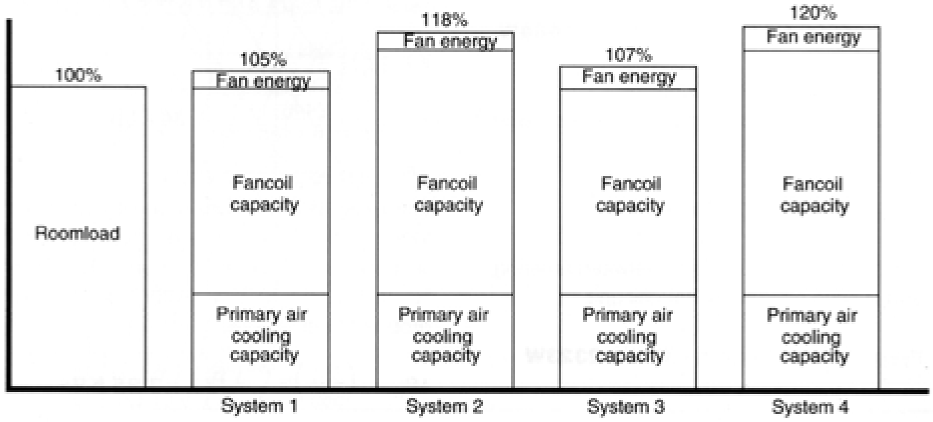
The most common methods to supply primary air are shown in the figures below:

Applications	**Schematic**
System-1: This is the most energy efficient system. Primary air is supplied directly into the room and the recirculation diffuser is ducted to the inlet of the fan coil unit. For this case the cooling energy available in the primary air can be deducted from the room load. An additional benefit with this configuration is the fan coil unit can be switched off in situations where primary air only is sufficient to maintain the required room conditions or when ventilation only is required.	
System-2: Primary air is supplied directly into the room and recirculation air is taken from the ceiling void. This is basically the same as system-1; however the cooling capacity must be higher because the temperature in the ceiling void is normally	

Applications	Schematic
higher than in the room.	
System-3: Primary air and recirculation air are both ducted to the intake of the fan coil unit. For this case, the entering air for the heat exchanger is lower in temperature and less humid than at system-1. The required cooling capacity from the fan coil unit is equal to system-1, however the air volume must be higher to supply both the primary and fan coil capacities of system-1.	
System-4: Primary air is ducted to the intake of the fan coil unit and recirculation air is taken from the ceiling void. This is basically the same as system-3; however the cooling capacity must be higher because the temperature in the ceiling void is normally higher than in the room.	

When primary air is supplied near the intake of the fan coil unit, part of the primary air will be lost directly into the exhaust system wasting conditioned air energy. This is wasting energy and should be avoided.

To demonstrate the difference in energy consumption, an analysis is indicated below:



Controls configurations

Automatic controls for the fan-coil unit generally comprise of an ON-OFF control to the fan motor, often in conjunction with three speed settings. A remote room temperature sensor is used to control both the fan motor and modulating valves on the heating coil and the cooling coil, and or three speed setting control to the fan motor.

Effectiveness and internal comfort

Fan-coil units provide excellent comfort levels for individual areas. With the temperature sensor located in the space, the heating or cooling provided by the fan-coil unit can be tempered according to the needs of the individual space.

Units with an outdoor air supply combine good temperature control with the ability to meet requirements for fresh air to be supplied to the space.

The fault most often identified in relation to fan-coil units relates to noise, as the fan is close to the occupied space. However, with a well-designed unit with a quiet fan and good sound attenuation this does not need to be an issue.

Maintenance and Balancing Considerations

The fan-coil unit is a free-standing unit and thus balancing is not required except if a ducted system is used to provide fresh air to the units. Balancing and commissioning of any ducted fresh air system may need adjustment, possibly affecting the output from the unit or the comfort in the space. The LTHW heating circuit or the chilled water circuit may need rebalancing or re-commissioning if indicated by poor performance of the unit in either cooling or heating mode.

Typically fan-coil systems have a large number of thermostats. Regular calibration of these thermostats is necessary.

If a fan-coil unit has a filter, this needs to be cleaned regularly as otherwise the air-flow may be reduced, which in turn affects the ability of the unit to properly heat and cool the occupied space.

Energy Saving Opportunities

The key areas for energy savings are:

1. Thermostats need to be calibrated regularly. In open-plan offices, poor calibration can cause adjacent units to "fight", with one unit heating and the other cooling.

2. Filters need to be cleaned regularly.

3. Fan-coils work well with hybrid ventilation systems. Interlocks can be arranged between the fan-coil unit in a space and the windows in the space, such that the unit switches off when the window is opened. This avoids the fan-coil unit heating or cooling the space when significant outdoor air is being introduced.

FAN COIL UNITS

System Pros	**System Cons**
1. Individual room temperature control. 2. Separate sources of heating and cooling for each space available as needed to satisfy a wide range of load variations. 3. Low distribution system space required as a result of reducing the air supply by	1. Limited to perimeter space. 2. The primary air supply is usually constant with no provision for shutoff. 3. Not applicable to spaces with high exhaust requirement.

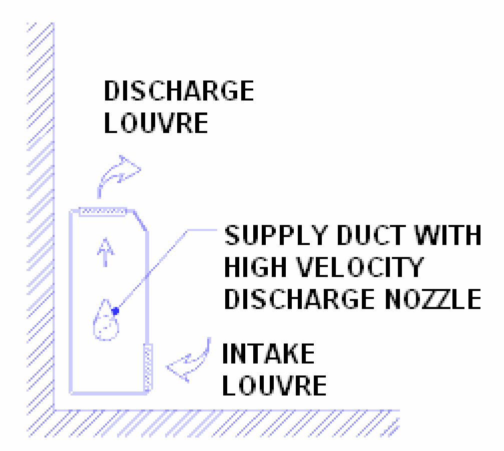
System Pros	**System Cons**
use of secondary water for cooling and high velocity air design. 4. Reduced size of central air handling equipment. 5. Dehumidification & filtration performed in a central plant room remote from conditioned space. 6. Outdoor air supply is positive. 7. Zoning of central equipment is not required.	4. Seasonal changeover is necessary.

INDUCTION UNITS

The inducting system is designed for use in perimeter rooms of multi-storey, multi-room building that may have reversing sensible heat characteristics. It is especially adapted to handle the loads of skyscrapers with minimum space requirements for mechanical equipment. Induction units are usually installed at a perimeter wall under a window. Some hotel rooms are provided with induction coils.

System description

Induction units are effectively a variant of a fan-coil unit. Instead of using a fan, primary air is injected into the unit at high velocity from nozzles, which induces a secondary air flow from the conditioned space - hence the name. The induced air flows over a cooling or heating coil to provide room temperature control.



Induction unit

In an induction system, ducted primary air is fed into a small plenum chamber where it is delivered through nozzles as high velocity jets. This induces secondary air from the room across the secondary coil. The right quantity of high pressure air is adjusted automatically in response to a thermostat located in the conditioned space. If the primary air alone provides sufficient temperature control, then an internal damper in the unit causes the induced secondary air to bypass the coil. Otherwise the induced air is heated or cooled through the coil.

Induction units are served by hot water and/or chilled water which is generated by central plant. Variations may take the form of two-pipe induction units, with a single cooling or heating coil, or may use an electrical heating.

Induction units are comparatively rare and not used in new installations.

Controls Configuration

Automatic controls for the induction unit comprise operation of the internal damper and/or heating and cooling coils according to a remote temperature sensor located in the space. The temperature of the primary air may also be varied according to the external ambient temperature or season.

Effectiveness and internal comfort

Induction unit installations can provide good comfort levels due to the individual room temperature control provided. Fresh air is also supplied to the space by the units.

Due to the increased air velocity from the primary air jet nozzles some noise may be generated by the induction units. With careful design this noise can be controlled to an acceptable level. The plenum is usually acoustically treated to attenuate part of the noise generated in the duct system and in the unit.

Energy saving opportunities

The energy savings opportunities for induction units are:

1. Room thermostats need to be calibrated regularly. Poor calibration may lead to overheating or overcooling which will waste energy.
2. Regular cleaning and adjustment of the primary air jet nozzles is important for the effective operation of the induction unit.
3. Air filters must be cleaned regularly

INDUCTION UNITS

System Pros	**System Cons**
In addition to the advantages of Fan-coil units, the induction units have 1. Minimal maintenance required as individual induction units have no moving parts, i.e. no fans 2. Without fans, these units provide a quieter space.	In addition to the disadvantages listed for fan coil units, induction units have 1. Installation requires the installation of both ductwork and pipework to the conventionally free-standing wall-mounted units. 2. Induction units can be complex to commission, when air balancing as well as heating and cooling water flows

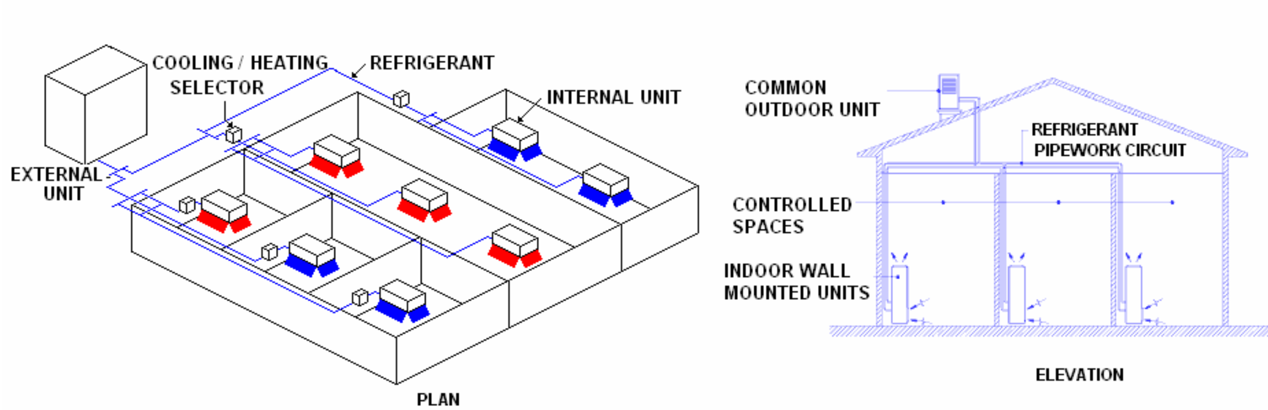
System Pros	**System Cons**
	must be correctly set. 3. The primary air supply is usually constant with no provision for shutoff. 4. Higher energy consumption due to increased power required by the primary pressure drop in the terminal units. 5. Controls tend to be more complex than for all-air systems. 6. Initial cost is usually higher than fan coil systems.

VARIABLE REFRIGERANT VOLUME SYSTEMS

In a VRV system, refrigerant runs around the building to individual fan-coils that can provide heating or cooling as required.

System description

Variable refrigerant volume (VRV) units comprise a number of room units which share a common refrigerant circuit, served from a common outdoor unit. Each of the room units can either heat or cool depending on how the refrigerant is used within the coil. Indoor units should be relatively near to one another, and the distance to the common outdoor unit kept to a minimum. The indoor units are similar in their operation to the split system air-conditioning unit.



Variable Volume Refrigerant Units

Controls Configurations

Automatic controls for the fan-coil unit generally comprise of on-off control to the fan motor, often in conjunction with three speed settings. A remote room temperature sensor is used to control both the fan motor and modulating valves on the heating coil and the cooling coil, and/or three speed setting control to the fan motor.

Effectiveness and Internal Comfort

VRV air-conditioning units provide good comfort levels, although noise made by the internal units may be noticeable. With the temperature sensor located in the space the heating or cooling provided by the fan-coil unit can be tempered according to the needs of the individual space.

VRV systems have the special feature that they can transfer heat around the building. Thus if a building requires heating in one area and cooling in another, the energy removed from the overheated area can in effect be transported to the cooler area. This can produce savings in buildings where heating and cooling requirements are diverse.

Maintenance and Balancing Considerations

VRV systems do not in general require balancing. Maintenance to the refrigeration system is required to ensure that their operation is satisfactory and that lifespan is maximized. The air filters on the intake grilles will require cleaning, according to the quality of the environment and the frequency of use of the units.

Energy Saving Opportunities

The key areas for energy savings are:

1. Regular maintenance to the refrigeration unit will improve both efficiency and effectiveness.
2. Cleaning of air intake filters is essential to maintain both performance and efficiency.
3. VRV units work well with hybrid ventilation systems. Interlocks can be arranged between the VRV air-conditioning unit in a space and the windows in the space, such that the unit switches off when the window is opened. This avoids the unit heating or cooling the space when significant outdoor air is being introduced.

DECENTRALIZED SYSTEMS

Decentralized systems are small air conditioner units that provide cooling only where needed rather than the entire space and these essentially cool small rooms. These are essentially direct expansion (DX) systems, which operate using direct expansion of refrigerant in the finned tubes across the air path.

Decentralized systems are often used as after hours support for centralized systems, allowing specific areas to be air-conditioned without requiring the operation of central plant. In such applications, both the central system and the decentralized units are sized to cool the particular conditioned space adequately without the other operating. In other applications, where decentralized units are added to supplement an inadequate existing system, they are selected and sized to meet the required capacity when both systems operate.

In smaller buildings, these are less expensive to operate than central units, even though their efficiency is generally lower than that of central air conditioners. Smaller room air conditioners (i.e., those drawing less than 7.5 amps of electricity) can be plugged into any 15- or 20-amp, 115-volt household circuit that is not shared with any other major appliances. Larger room air conditioners (i.e., those drawing more than 7.5 amps) need their own dedicated 230-volt circuit.

The major types of decentralized systems are:

1. Split Systems
2. Window Units
3. Heat Pumps

All these comprise of a hermetic sealed compressor/s, evaporator (cooling coil fabricated out of copper tubes and aluminum fins), a supply air fan, filter and a condensing unit. These are essentially the factory assembled self-contained units and are also known as local systems.

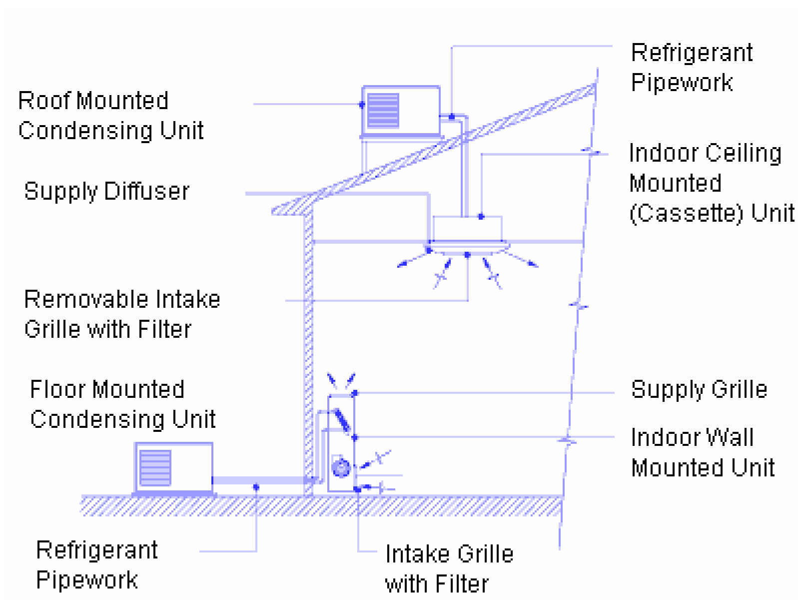
SPLIT SYSTEMS

As the name suggests, the split systems are individual systems in which the two heat exchangers are separated (one outside, one inside). The indoor unit (evaporator) consists of a direct expansion (DX) cooling coil, filter and fan whereas the outdoor unit (condensing unit) contains one or more reciprocating/rotary compressors, a condenser and a fan. The evaporator and the condensing unit are connected by refrigerant piping.

System description

Split system air-conditioning units comprise internal and external units linked by refrigerant pipework. The refrigerant compressor, installed in the outdoor unit, pumps the refrigerant through the indoor unit and the outdoor unit arranged in closed loop. The refrigerant picks the heat from the indoor unit (evaporator coil), and rejects energy to the outside atmosphere as it goes through the outdoor unit (condensing coil). Energy rejected is the sum of the energy taken indoors plus the energy consumed by the compressor in pumping the refrigerant through the refrigerant circuit.

The indoor unit fan pulls or pushes air around the outer surfaces of the coil inside the indoor unit, taking warm air from the room and injecting cooled air into the room in summer. The air passing through the indoor unit is cooled; say to 60°F, before re-circulated back to the room.



Split system air-conditioning units

Indoor units can be wall mounted or ceiling mounted. Recessed ceiling mounted units are often referred to as "cassette" type units. Refrigeration pipework will comprise liquid and gas pipelines, of unequal sizes, and may be restricted in length, requiring careful location of the indoor and outdoor units for larger buildings. The outdoor unit can be roof mounted, wall mounted or fixed directly to the ground.

Split unit is a choice in applications, which require interior zone cooling. Remember, a window unit require outside exposure for heat rejection and cannot be used for interior rooms.

Controls configurations

Split system air-conditioning units provide good individualized control. A hand held remote controller is often provided, linked to a wall unit with thermostat temperature control. The fan of the indoor unit may have two or three speeds to give some modulation of heat or cooling output. The split system may be reversible, operating as a heat pump to provide heating to the space. Alternatively an electric heating element is used to provide heating.

With the temperature sensor located in the space the heating or cooling provided by the fan-coil unit can be tempered according to the needs of the individual space.

Maintenance and balancing considerations

Split system air-conditioning units are free-standing and independent and thus balancing is not an issue. Maintenance to the units is required to ensure that their operation is satisfactory and that lifespan is maximized. The air filter on the intake grille will require cleaning, according to the quality of the environment and the frequency of use of the units.

The use of split systems for large areas can cause excessive maintenance costs due to the large number of moving parts - each split system has its own compressor and a sealed refrigerant system.

WINDOW UNITS

Window-mounted air conditioners cool the individual conditioned spaces. A window unit is an encased assembly designed primarily for mounting in a window, through a wall, or as a console. These units are designed for comfort cooling and to provide delivery of conditioned air to a room without ducts. They include a prime source of refrigeration, dehumidification, means for circulating and cleaning air, and may also include means for ventilating, and/or exhausting and heating.

They are cheap and easy to install, being fitted into windows or through walls. They are quite common in small, mid-grade suburban office space, older institutional buildings and accommodation blocks.

They have a low initial cost and are quick and easy to install.

System description

Window units are simple refrigerate coolers packaged into a single box that produces cool air on one side and rejects hot air on the other. These require outside exposure for heat rejection and cannot be used for interior rooms. Split unit option shall be used in such places. The refrigerant compressor now is part of the machine locating at the window area. Since this compressor gives out most noise, among other components, the window unit will make the room acoustically inferior to other air conditioning systems.

Controls configuration

The controls for window units are simple and inbuilt, with a rotating switch marked with a hot-cold scale with no temperature settings. Most units will heat as well as cool. Fresh air control, if it exists, is normally adjusted by a manual lever. Fresh air exchange for the room can be provided by: -

- Setting the “ventilator” switch of the window air conditioner to “open” position
- Installing a ventilating extract fan in the room to extract room air to outside – caution- not to oversize the fan
- Naturally leaking of air in and out of the room

Maintenance and balancing considerations

Window units are often poorly maintained. They are not an engineered solution to air-conditioning needs, but are often used where air-conditioning is either an afterthought or not intended for regular or long term use. The units require regular servicing of the refrigeration system on a regular basis to operate, and the cleaning of filters. The manual levers for fresh air-control sometimes become casualties to age and rough handling, which can lead unwanted fresh air flow. There are no balancing considerations for this system.

HEAT PUMPS

A heat pump is a device that acts as an air conditioner in the summer and as a heater in the winter. Heat pumps look and function exactly like an air conditioner except it has a reversible cycle. It contains a 4-way reversing valve that lets it switch between "air conditioner" and "heater."

A cooling only system cools the indoor air but a heat pump provides cooling in summer season and heating in winter season. A supplementary electric resistance heater may also be used to assist the heat pump at lower outdoor temperatures. In colder climates, heat pumps require a defrost period. During defrost times the electric heater is the only means of heating the interior of the building. These units are manufactured as either split or packaged systems.

Heat pumps for air conditioning service may be further classified as:

Air-to-Air Heat Pumps

The air-to-air heat pump is the most common type of heat pumps. It is particularly suitable for factory-built unitary heat pumps, and has been widely used for residential and commercial application. Air is used as the heat source and heat sink. Extended surface, forced convection heat transfer coils are normally employed to transfer the heat between the air and the refrigerant. When selecting or designing an air-source heat pump, two factors in particular must be taken into consideration:

1. The variation in temperature experienced in a given locality.
2. The formation of frost

Water-source Heat Pumps

The water-source heat pump uses water and air as the heat source or heat sink depending on the mode of operation. When cooling, water is used as the heat sink, and the heat pump operates as a water-cooled air conditioner. When heating, water is used as the heat source and the equipment operates as a water chiller.

The water-source heat pump is suitable for many types of multi-room buildings, including office buildings, hotels, schools, apartment buildings, manufacturing facilities and hospitals.

Heat Pump Efficiency

Because heat pumps move heat rather than generating heat, these can provide up to 4 times the amount of energy they consume. If you heat with electricity, a heat pump can trim the amount of electricity you use for heating by as much as 30% to 40%. High-efficiency heat pumps also dehumidify better than standard central air conditioners, resulting in less energy usage and more cooling comfort in summer months.

The measurement of heating efficiency of a heat pump is the coefficient of performance (COP). It is the ratio of heat output to electricity input using the same units (BTUH or kW). A heat pump delivers from 1½ to 3½ units of heat for every unit of electricity it uses; saves from 30 to 60 per cent electric heating bills, depending on the geographic location and equipment used. COP of a heat pump is generally greater than 1. Heat output of a heat pump, in terms of BTUH per kWh, is a product of its COP and a factor of 3400 (e.g. if the COP of a heat pump is 1, then its heat output would be 1x 3400=3400 BTUH per kWh).

The heating efficiency of a heat pump may also be measured using a rating known as HSPF (Heating Season Performance Factor). This rating is obtained by multiplying the COP of a heat pump by a factor of 3.4.

<u>Advantages</u>

1. Affords opportunity for energy conservation by recovering heat from interior zones and/or waste heat and by storing excess heat from daytime cooling for night time heating
2. No wall openings required.
3. Longer expected life than air-cooled heat pumps.
4. Lower noise level because condenser fans are eliminated.

5. Energy for the heat pumps can be metered directly to each tenant.

6. Total life cycle cost frequently compares favorably to central systems when considering relative installed cost, operating costs, and system life.

Disadvantages

1. Space required for boiler, heat exchanger, pumps and heat rejecter.

2. Higher initial cost than for most other multiple-packaged unit systems.

3. Reduced airflow can cause the heat pump to cycle cutout. Good filter maintenance is imperative.

SECTION -2 HVAC SYSTEM COMPONENTS

A heating, ventilation and air-conditioning (HVAC) system may be defined as an assembly of components with a particular structure and a defined function. There are literally dozen or hundred of ways in which basic HVAC components may be assembled into systems. In the following pages we review the operation and use of a range of components.

Chillers

Chillers are a key component of most centralized air-conditioning systems. The function of a chiller is to generate chilled water, which is distributed to large spaces for cooling. The two principal types of chiller are air-cooled and water cooled. Compared to water, air is a poor conductor of heat and therefore air-cooled chillers are larger and less efficient. The typical condensing temperature for an air-cooled chiller is 120°F as opposed to a 105°F in a comparable water condensed chiller. Air-cooled chillers also operate at higher compressor ratios – which mean less cooling per watt energy consumption.

Air cooled chillers are generally located outside the building and reject heat directly to the atmosphere, while water cooled chillers are generally located within the building and use cooling towers located outside the building to reject the heat.

<u>Circumstances favoring Air-cooled Systems</u>

Air cooled chillers are favored over the water cooled systems under following circumstances:

1. Where water is scarce or quality water is not available
2. Where the system is not required to operate 24 hours.
3. Where the system is not to be located in or around noise restricted areas
4. Where there is adequate and accessible roof top or ground space for the system equipment
5. Where sitting of cooling tower is restricted due to Legionella risk minimization constraints.
6. Where air-conditioning requirement is less than 200 TR

7. There may be statutory requirements for health and safety that may not permit use of cooling towers in certain areas.

8. A high humidity climatic condition in the tropical areas where the effectiveness of the cooling towers is significantly reduced.

Circumstances favoring Water-cooled Systems

Water-cooled chillers are generally favorable over the air-cooled systems under the following circumstances:

1. Where the system is required to operate 24 hours.

2. Where there is limited roof top or ground space for the system equipment

3. Where noise minimization and aesthetics are of relative importance

4. Larger system capacity requirement typically above 200 TR.

The present trend leans towards the use of air-cooled condensers. Results from recent generic studies on comparative life cycle costs of air cooled and water cooled systems indicate that each system is considered to be more favorable than the other over a certain range of plant capacity. As a guide, conclusions could be generalized and summarized as follows:

Capacity Range (TR)	**Favorable System**
40 to 200	Air-cooled chilled water system (explore the pros and cons of using multiple DX systems if possible)
200 and above	Water-cooled chilled water system

Chiller Types

Different types of chiller are also used depending upon the type of compressor used as part of the refrigeration circuit. The different types of compressor are as follows:

1. **Reciprocating:**

   Reciprocating compressors are categorized as positive displacement machines. These are available in two basic types: hermetic sealed units and units of open construction. In hermetic sealed units, the motor and the compressor are direct-coupled and housed in a single casing that is sealed to the atmosphere. In open construction units, the motor and the compressor are in separate housings. In general, open construction units have a longer service life, lower maintenance requirements and higher operating efficiencies. The hermetic sealed units are most common particularly in small capacities. Single stage reciprocating machines have an ability to operate at compression ratios of 10 to 12.

   The capacity control in reciprocating machine is achieved through 'On-Off' or 'Loading-Unloading' of compressor cylinders.

   Reciprocating machines are manufactured in capacities from 0.5 to 150 TR.

   The main factors favoring reciprocating machine is low cost. The other advantage is that multiple reciprocating machines can be installed to closely match the building loads. Multiple units allow flexibility to operate machines per the need. If properly managed this could attribute to significant energy savings during low loads.

   A major drawback is a high level of maintenance requirement's, noise and vibration. Since the capacity is limited to 150 TR, multiple units cost more than other options. Multiple chiller configurations require large space and consume more energy per ton of refrigeration.

2. **Rotary Screw:**

   Rotary or screw chillers, like reciprocating machines are positive displacement compressors. Rotary is a wider term that may include vane, eccentric, gear or screw types. The commercial refrigeration installation rely more on screw machines. Screw compressors are available in several designs, both single screw and twin screw, with oil-free and oil-injected designs in both types. Twin-screw oil-injected compressors are slightly more energy efficient at moderate compression ratios. Twin-screw compressors have an ability to operate at compression ratio of 30. Units are available in both hermetic sealed and open construction.

Screw compressors are used in the mid-range of unit sizes, around 20-1000 tons. They are compact and have less moving parts, hence lower maintenance costs and longer life spans. Continuously variable loading can also be provided, improving partial load efficiencies.

The capacity control in screw compressor is achieved thorough a moveable slide stop valve, which will vary the compressor internal volume ratio to achieve optimum energy consumption during part load operation.

The major drawback is their high cost. For smaller loads, reciprocating machines are less expensive to purchase and for large loads centrifugal machines cost less.

3. **Rotary Scroll:**

Scroll compressors have been used in commercial practice for systems that have capacity less than 30 TR. Scroll compressors are used in smaller units such as unitary heat pumps, and may be up to 10% more efficient than the equivalent sized reciprocating unit. On such small sizes, these do not affect the life cycle economics drastically.

4. **Centrifugal:**

Centrifugal chillers are categorized as variable volume displacement units. Like reciprocating machines these are also available in both hermetic and open construction. Commercially the hermetic sealed units are more widely used, despite its lower operating efficiency. Centrifugal chillers for refrigeration applications are generally designed for a fixed compression ratio of 18.

The capacity control is achieved through the use of inlet vanes on the impellers that restrict refrigerant flow.

The centrifugal chillers are manufactured in capacities from 90 to 2000 tons and are generally used for capacities above 200 tons.

The main factor favoring centrifugal machine is their high operational efficiency at full load, compact size and availability in large sizes. The biggest drawback of centrifugal machine is a very poor part load performance and inability to operate at low cooling loads. At extreme low loads, these chillers are prone to a condition known as surging.

Chiller Efficiency Terms

There are a number of ways to express the efficiency of a heating or cooling source.

1. COP – "Coefficient of Performance" - is the measure chiller efficiency measured in Btu output (cooling capacity) divided by Btu input (electric power). Typical values are 2 – 4.

    - Cooling capacity is specified in tons of refrigeration; 1 ton is equivalent to 12000 Btu per hour.

    - 1 kWh of electric power is equivalent to 3412 Btu per hour; multiplying the COP by 3.412 yields energy efficiency ratio.

    A chiller that produces three units of cooling to one unit of electricity is said to have a "coefficient of performance" or "COP" of three.

2. SEER – "Seasonal Energy Efficiency Ratio" – Total cooling output of air conditioning equipment during normal operating season (in Btu per hour) divided by the total electric input during the same period in watt-hour. SEER ratings may range from less than 5 to more than 14. It applies to units of less than 65,000 Btu per hour capacity

3. EER – "Energy Efficiency Ratio" – Equipment cooling capacity in Btu per hour divided by total energy input in watts at full-load conditions. The power input includes all inputs to compressors, fan motors, and controls. EER is always greater than one; typical values are 8 – 10. It typically applies to larger units over 65,000 Btuh capacity.

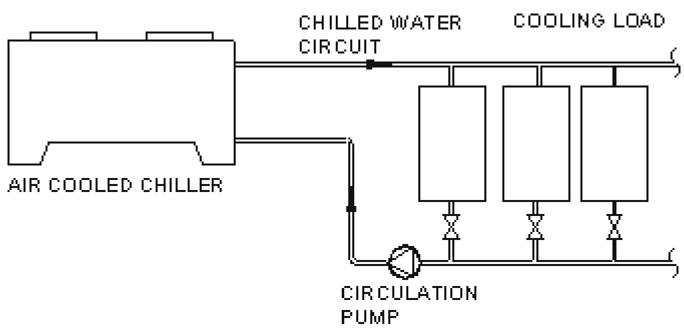
4. IPLV – "Integrated Part Load Value" – A single number value that expresses part load efficiency of air conditioning equipment (based on EER or COP) weighed by operation at various part load capacities.

Circuit Arrangements

The following examples of chiller water circuit arrangements are included to indicate how chillers are used to meet building cooling loads.

1. Air cooled chillers
2. Water cooled chillers, open circuit cooling tower
3. Water cooled chillers, closed circuit cooling tower
4. Air cooled chiller, ice storage option
5. Air cooled chiller, free cooling option

Air-cooled chillers

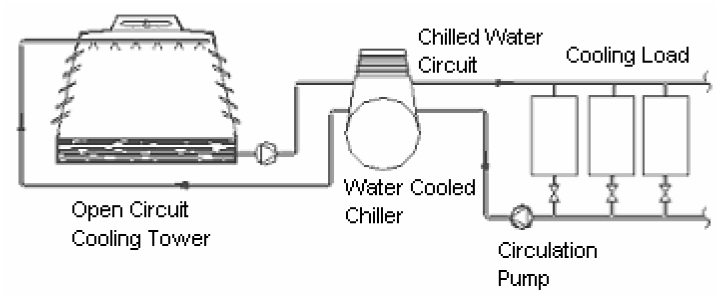


Air cooled chillers

Air-cooled chillers are effectively a single package system, into which chilled water is directly plumbed.

Note that centrifugal chillers require water-cooled condensers. Air-cooled condenser is not a recommended option for the centrifugal machines operating at low-pressure refrigerants. The screw and reciprocating machines are available in both air-cooled and water-cooled condenser options.

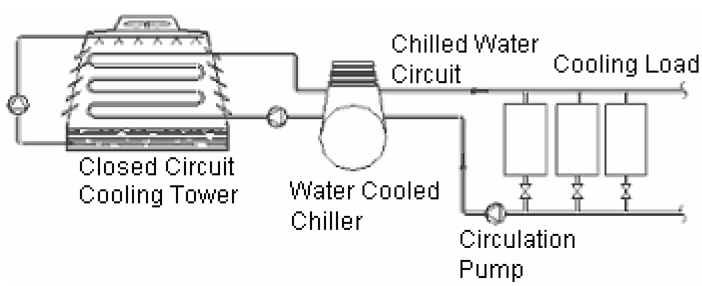
Water cooled chiller - open circuit cooling tower



Water cooled chiller, open circuit cooling tower

Water cooled chillers have a separate flow of water passing through their condenser, typically at around 20-30°C. This is then passed through a cooling tower. In an open circuit cooling tower, the condenser water is directly cooled by spraying it onto large sheets called media and passing air through the media.

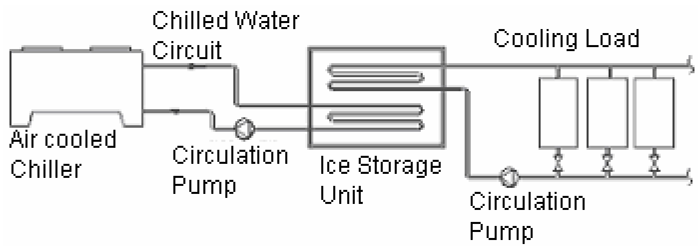
Water cooled chiller - closed circuit Cooling Tower



Water Cooled Chiller, Closed Circuit Cooling Tower

A closed circuit cooling tower has its own flow of water that is sprayed onto the condenser water circuit pipes to cool them down indirectly.

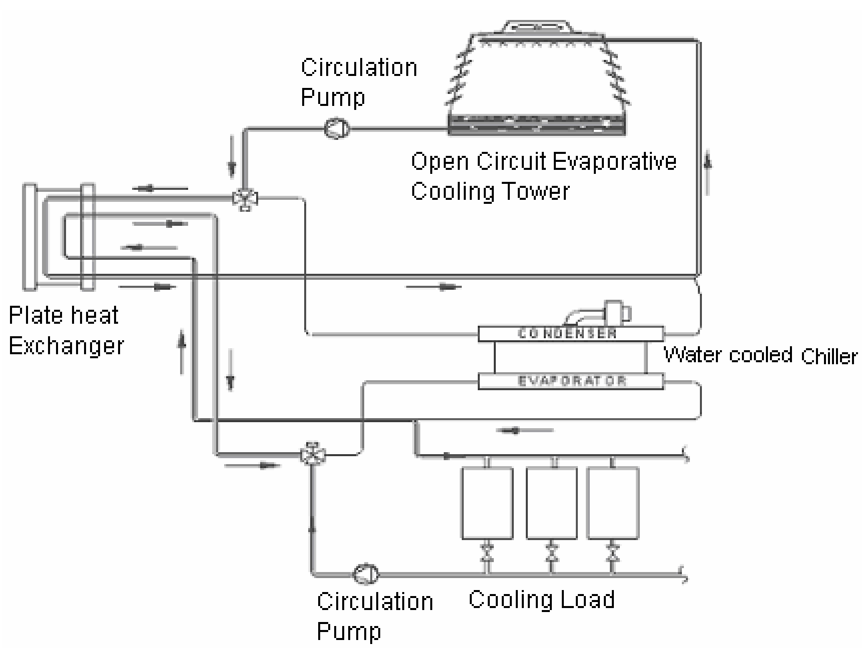
Ice storage option



Air cooled chiller, ice storage option

Ice storage may be used to reduce chiller size for variable cooling load profiles or to take advantage of night electricity rates. Ice storage systems can use water cooled or air cooled chillers.

Free cooling option



Air Cooled Chiller, Free Cooling Option

During certain conditions it may be possible to provide cooling, without the operation of a chiller, using water circulated directly through the cooling tower.

Energy Efficiency Opportunities

There are a number of opportunities to improve energy efficiency with chiller systems. Here are some of the best.

1. **Chiller Sequencing -** In general, chillers are installed in groups, which allow the number of chillers operating to be adjusted to meet the load. This is important because the efficiency of chillers decreases as the load gets smaller - running a 500kW chiller on 50kW load will waste a lot of energy. The adjustment of chiller staging can produce significant performance benefits.

   The applications where cooling is required for critical service delivery, the relevant areas need to be classified as essential. For essential services continuity of supply is critical, therefore the provision of cooling cannot be relied on a single chiller. One back up unit would be required. The provision of 1 additional back up unit is known as N+1 strategy. This strategy involves the provision of one more unit than the quantity required for the normal system operations. For example if the total capacity requirement is 1500 TR, 2 units each of 750 TR should be provided instead of one unit of 1500TR if the essential load is less than 50% of the total load. Otherwise 2 units having the capacity equals to the essential load shall be required. A life cycle cost economics of using 3 X 50% option could also be investigated.

2. **Operating Temperatures -** The wider the temperature difference between the condenser and evaporator of the chiller (the hot and cold sides, respectively), the poorer the efficiency. Thus if you can raise the chiller water temperature for some of the year and decrease the condenser water temperature for some of the year, this will improve efficiency.

3. **Operational regimes:** It is imperative to determine which portion of the total load required 24 hours operations. Say in an office building operating for 10 hours a day and the data center room requiring 24 hrs operations, it shall be prudent to use multiple chillers with capacity matching nearly the critical essential load for overall economics.

   It would be more cost efficient to operate a smaller capacity chiller at full load than a bigger capacity at part load.

4. **Suction pressure:** While designing a refrigeration system, a specific suction pressure is usually specified, and often time's designers forget that this may change dramatically. Lower suction pressure signify lower evaporator temperature, which is the refrigerant temperature required to absorb heat from the medium being cooled. Lowering the evaporator temperature 10 deg from a base of 40°F and 105°F reduces the capacity about 24%, and at the same time increases the compressor HP per ton about 18%. The twin-screw compressor allows it to handle large variations in suction pressures with small changes in operation efficiency. A reciprocating compressor can also accommodate variations. With the centrifugal compressor, there is little potential variation available and even a small variation can cause considerable problems. In simple words the screw and reciprocating machines can meet additional capacity requirements caused due to expansion or modification just by control settings.

5. **Discharge Pressure:** Higher discharge pressure signifies higher condensing temperature, which is the refrigerant temperature required to reject heat to the condensing medium. Increasing the condensing temperature 15 deg from a base of 40° F and 105°F reduces the capacity about 13% and increases the compressor HP per ton about 27%. Discharge pressure is of considerable importance in refrigeration systems, particularly in the water-cooled condenser. The lower the temperature of discharge gas, the less fouling with in the condenser. In screw compressor, the discharge temperature of refrigerant gas is held to a minimum due to the lubricant cooling system. The highest system pressure occurs in reciprocating compressor while in centrifugal compressor it is kept low due to inter-stage cooling.

6. **Cooling load profiling:**

    - Peak load demand determines the overall capacity of the system. The total chiller capacity in tons of refrigeration shall match or exceed the peak building load.

    - Part load requirements determine the number and size of chillers required. Cooling load profile will help to determine the type of chiller to use and if single or multiple chillers should be installed. Multiple chiller installations allow facilities professionals to stage their operation to match building loads while keeping the chillers operating at energy efficient loading. With high capacity centrifugal chillers part load application should not be less than 20 to 35% of the full load capacity. If the part load requirement is less than this amount, a reduced capacity chiller should be selected.

- Essential service requirements determine the system reliability requirements. Say in a hospital, the Operation Theater or infectious area call for controlled environment all the time. Therefore the selection of chiller should take into account a dedicated chiller capacity to meet the essential loads.

7. **Cooling Tower Sequencing -** Chillers are normally attached to a bank of cooling towers. As the chiller load varies, so does the need for cooling towers. Adjusting the sequence of cooling tower operation can save energy.

8. **Chiller Heat Recovery -** If your building has a need for hot water, you can recover approximately 10% of the heat rejection from a chiller as hot water at 140-160°F. This can avoid unnecessary boiler operation, and thus save energy. The remainder of the heat is only available at 85^o F or less and this is little use for anything other than perhaps swimming pool heating.

9. **Improved chillers:** The Montreal Protocol, intended to protect the ozone layer, has begun a phase-out of CFC (chlorofluorocarbon) refrigerants. This has triggered a wave of chiller replacements that should accelerate during the next 10 years. Energy-efficiency measures in conjunction with chiller retrofits can reduce the load and the necessary chiller size, making the change more economical overall. New technologies such as direct digital controls and variable frequency drives, combined with improved design, maintenance and operation, can decrease chiller energy consumption by more than half while improving their reliability.

10. **Environmentally Friendly Refrigerants:** AC refrigerants come in many varieties. R-22 is the most common, however, due to interactions with the ozone layer R-22 is being phased out. Refrigerants manufactured as replacements for R-22 are HFC-134a, R-410a, R-410b to name a few. The new refrigerants do not contain the chlorine atom and are not harmful to the earth's ozone layer.

CONDENSER & CONDENSING UNITS

Condensers can be put in the following groups: water-cooled, air-cooled, and evaporative. The term "evaporative," when used in reference to the condensing process, refers to the cooling effect brought on by the natural evaporation of water exposed to air currents. The heat rejected by a condenser comes from the heat absorbed by the evaporator and the heat of compression that is added by the compressor. Since the compressor and condenser work as one to compress and condense refrigerant vapor, these two parts, when combined into one package, are called a condensing unit.

1. **Shell & Tube Condensers -** Shell-and-tube condensers are used for most of the water-cooled refrigeration systems. The shell-and-tube type of water-cooled condenser is like the direct expansion water chiller. But most shell type water-cooled condensers have the cooling water flowing inside the tubes, and the refrigerant that it condenses is inside the shell, but outside the tubes.

2. **Air Cooled Condensers -** Air-cooled condensers are most popular in areas where water is in short supply, where there is a costly water supply, or where the use of water for air conditioning is restricted at times. They also find wide use in those jobs where low maintenance is a prime need. In addition, air-cooled condensers are used in a lot of installations, because they keep down the cost and do away with the installation of water pipe. They also do not require drainage to keep them from freezing in areas where the climate changes a lot and the cooling system must be turned on and off several times in a year. Propeller fan air-cooled condensers use a fin-and-tube coil like the coil described in the Direct-Expansion Coils section. The refrigerant vapor is condensed inside its tubes by giving up heat to air which flows across the coils. These condensers are, in most cases, placed outdoors, or at least the air is taken to the outdoors.

COOLING TOWERS

Water cooled chillers require a source of cooling water, such as cooling tower water, to extract heat from the refrigerant at the condenser and reject it to the ambient environment. Cooling Towers for HVAC duty are usually described by their tons of cooling capacity. The cooling capacity indicates the rate at which the cooling tower can transfer heat. One ton of cooling is equal to 12,000 BTUs (British thermal units) per hour, or 200 BTUs per minute. The heat rejected from an air conditioning system equals about <u>1.25 times</u> the net refrigeration effect. Therefore the equivalent ton on the cooling tower side actually rejects about 15,000 Btu/hour (12000 Btu cooling load plus 3000 Btu's per ton for work of compression). Cooling tower capacities at commercial, industrial, or institutional facilities typically range from as little as 50 tons to as much as 1,000 tons or more. Large facilities may be equipped with several large cooling towers.

The controlling principle of a tower system is water's inherent nature to lower its own temperature as it evaporates. By evaporating a small part of the process water, the temperature of all process water is lowered.

Tower cells accomplish this by spraying fine water droplets in a contained environment. The droplets fall through a stream of upwardly moving air. The more contact time of the air and water, the greater the amount of evaporative and heat transfer. To significantly increase the amount of contact time, cells include "fill" material to reduce the free falling of water and enlarge the surface area of water to air. The result is greater exposure of water to air. With an increase in exposure, there is a corresponding increase in cooling capacity.

Air must absorb water for evaporation to occur. The higher the level of humidity, the less air is able to absorb water and, as a result, the less efficient the tower system in cooling. Typically, cooling tower systems capacity are rated to lower 95°F water to 85°F at 78°F wet bulb. Wet-bulb temperature of the air is the lowest temperature possible for evaporation due to ambient or surrounding environment so the temperature of the water cannot drop below the prevailing wet bulb temperature of the air.

Each tower system must be specifically sized for each geographic area's prevailing summer wet bulb temperature. While some geographic areas may experience cold climates, a tower's cooling capability is usually set at no colder than 70°F during winter months. High efficiency mechanical draft towers cool the water to within 5 or 6°F of the wet-bulb temperature, while natural draft towers cool within 10 to 12°F.

Types of Cooling Towers

There are three basic types of towers.

1. **Forced Draft Tower** - In forced draft cooling towers, air is "pushed" through the tower from an inlet to an exhaust. A forced draft mechanical draft tower is a blow-through arrangement, where a blower type fan at the intake forces air through the tower. A

forced draft tower has a sensor that monitors the process water temperature after it exits from the tower. The fan engages or disengages when the process water temperature rises either above or below the desired set point.

2. **Induced Draft Tower** - A second type of tower, induced draft has a fan in the wet air stream to draw air through the fill. Induced draft cooling towers are characterized as Cross-flow and Counter-flow designs, by virtue of air-to-water flow arrangement. The difference lies in the FILL arrangement. In a counter-flow induced draft cooling towers, air travels upward through the fill or tube bundles, opposite to the downward motion of the water. In cross-flow induced draft cooling towers, air moves horizontally through the fill as the water moves downward. An induced draft mechanical draft tower is a draw-through arrangement, where a fan located at the discharge end pulls air through tower.

3. **Natural Draft Tower** - A third type, natural draft tower, has no mechanical means to create airflow. Natural-draft cooling towers use the buoyancy of the exhaust air rising in a tall chimney to provide the draft. Warm, moist air naturally rises due to the density differential to the dry, cooler outside air.

In all types, towers use the force of gravity to drain water into a sump from which the pump delivers the water to the chiller condenser where it picks up heat of the refrigerant. The now-warmed water continues to flow back to the outdoor tower through return lines. The cycle continuously repeats.

BOILERS

Large commercial buildings use a central heating system that pumps hot water for heating to a number of buildings. The system makes use of hot water or steam boiler to produce heat. The production of heat is a relatively straightforward affair; typically, all a boiler does is to burn fuel and produce heat. This heat is absorbed by water, which flows through tubes inside the boiler.

Gas boilers are the most common type in buildings HVAC, although occasionally you will find electric boilers or oil boilers. You can also sometimes find dual fuel boilers, typically able to use gas or oil.

Most boilers operating in buildings produce hot water in the region of 70-85°C for the purpose of providing a heat source for the air-conditioning system. These boilers may also serve the domestic hot water systems, but these are more often served by separate hot water cylinders.

Boiler Types

The common type of boilers for HVAC installations include:

1. **Cast Iron Boilers** - These boilers range up to 10 million Btu/h and are most typically used in residential or light commercial heating applications and can be retrofitted with a domestic heat coil for domestic hot water use. For heavier commercial use several of these boilers can be set up on a manifold piping system and offer step-up sequencing to meet higher demands when needed. Cast iron boilers can provide either steam or hot water heat in low pressure applications. Combustion efficiencies range from 75% to 93% depending on (whether it is steam or hot water) and combustion controls, flue dampers, sequencing of multi-stage set-ups, frequency of tune-ups and/or air and/or water pre-heaters (economizers).

2. **Water Tube Boilers** - From 10 million BTU/h up to 300 million Btu/h, these boilers are generally found in medium to large commercial/industrial use and can be either steam or hot water in low to high pressure applications. They can be either oil, coal, or gas fired and pass hot flue gases around tubes filled with water. Combustion efficiencies vary depending on several factors including: whether it is steam or hot water, combustion controls, flue dampers, frequency of tune-ups and/or air and/or water pre-heaters (economizers).

3. **Fire Tube Boilers** - from .6 million BTU/h up to 50 million BTU/h these boilers use hot flue gases passing through tubes submerged in water generally found in medium to large commercial/industrial use and can be either steam or hot water in low to medium pressure applications. Again as with the water tube boiler combustion efficiencies depend on several factors as noted above.

One other type of boiler not mentioned above is the **electric boiler**. One could say that an electric hot water heater is an electric boiler although there are electric boilers that can heat water to steam temperatures.

Boiler Efficiency Measurement

When we talk about boiler efficiency we talk about two related types of efficiencies and both can have an impact on the energy budget. There is boiler efficiency and there is combustion efficiency. A decrease in combustion efficiency will decrease boiler efficiency but not necessarily vice versa.

1. **Combustion Efficiency** assesses the efficiency with which the fuel is being burned, and is a good measure of the tuning of a boiler. It is measured by dividing the usable heat produced by the fuel input in BTU/h content. This calculation is based on the actual heat available produced by the system after heat loss up the stack and other heat losses which do not provide usable heat. Excess air is partially responsible for the heat loss but it is necessary to complete the combustion process. Therefore it is important that the burner system is tuned and monitored on a regular basis. Using combustion analyzers the O_2 and stack temperatures can be monitored for spikes which will alarm the maintenance crew about possible problems. It is important to maintain good combustion efficiency for overall boiler efficiency.

2. **Boiler Efficiency** assesses the ratio of heat out to fuel in, and is the best overall measure of boiler operation, accounting for boiler load variation. It is measured by dividing combustion losses, radiant heat losses from the boiler jacket and near boiler piping, and unknown losses (losses from tube scaling, soot build-up (exchanger thermal efficiencies)) by total fuel input in BTU/h. Much of boiler efficiency is determined by combustion efficiency and a lot of maintenance departments focus on combustion efficiencies and ignore the other losses.

3. **Thermal Efficiency** is related to how efficient the heat exchanger is working. Things like soot build-up or water scaling can reduce the efficiency of a boiler.

4. **Steady-State Efficiency** is the efficiency of the boiler running full blast under maximum load.

5. **Overall Seasonal Efficiency** is important to track to see if there is an annual degradation in efficiency. It gives you the big picture from year to year for comparison.

To get the most out of your boiler system it is necessary to implement a complete plan to maintain every aspect of boiler efficiency. Providing that a boiler maintenance/efficiency plan is put into place and qualified boiler technicians perform the boiler tuning with the appropriate tools a cost savings in energy use can be realized.

Energy Efficiency Opportunities

Heating technologies for commercial buildings have become more efficient. An important consideration is proper size; an efficient heating system size should run constantly at full load on the coldest day the building is designed to handle. Although designers sometimes oversize a heating system to provide a margin of error, an oversized system will run inefficiently at partial loads through the heating season

There are several major energy efficiency opportunities for boilers:

1. **Boiler sizing:** It is relatively common practice in building design to oversize the boilers. However, the efficiency of a boiler drops as the loading decreases, although above 50% loading most boilers will still perform reasonably well. Below 25% loading major energy savings may be possible by installing another, smaller boiler.

2. **Boiler sequencing:** Linked to the issue of sizing is the issue of sequencing of multiple boilers sets. The ideal sequence minimizes the number of boilers working at part load and maximizes the load on all boilers to get the best efficiency result.

3. **Boiler control tuning:** The control of boilers in terms of the amount of air provided for combustion is critical to the efficiency of the boiler.

AIR HANDLING UNITS

An air handling system is a means of providing conditioned air to the space in order to maintain the environmental requirements. While there is no "one correct way" to design an air handling system, all air handling systems have basic components that are used. To meet the whole sensible and latent cooling needs of a space with the cool air they supply, these systems use air handling units and, in most cases, chilled water coils to give cooled and dehumidified air to the space. These systems may use low-, medium-, or high-pressure air distribution systems. All air systems are subdivided into single -zone, multizone, dual-duct, reheat, and variable air volume systems.

1. **Single-zone systems -** Single-zone systems serve just one temperature control zone and in most cases, these are controlled by varying the quantity of chilled water or refrigerant, adding reheat, adjusting face or bypass dampers, or a combination of these.

2. **Multizone systems -** Multizone systems are used to serve a small number of zones with just one central air handling unit. The air handling unit for multizone systems is made up of heating and cooling coils in parallel to get a hot deck and a cold deck. For the lowest energy use, hot and cold deck temperatures are, as a rule, automatically changed to meet the maximum zone heating (hot deck) and cooling (cold deck) needs. Zone thermostats control mixing dampers to give each zone the right supply temperature. Not much in use in modern air-conditioning systems.

3. **Dual-duct systems** - Dual-duct systems are much like multizone systems, but instead of mixing the hot and cold air at the air handling unit, the hot and cold air are both brought by ducts to each zone where they are then mixed to meet the needs of the zone. It is common for dual-duct systems to use high-pressure air distribution systems with the pressure reduced in the mixing box at each zone. Not much in use in modern air-conditioning systems.

4. **Reheat systems -** Reheat systems supply cool air from a central air handler as required to meet the maximum cooling load in each zone. Each zone has a heater in its duct that reheats the supply air as needed to maintain space temperatures. Reheat systems are quite energy-inefficient and have become rare in new buildings.

5. **Variable air volume systems –** Variable air volume system controls the temperature in a space by varying the volume of supply air rather than varying the supply air temperature. Zone airflow is modulated by individual variable air volume (VAV) boxes serving respective zone whereas total system airflow is varied by the use of inlet vanes, discharge dampers, speed control, and variable pitch blades. This is most common system in commercial office buildings.

Air handling system major components

Air handling units may consist of a supply fan and coil section with a chilled water or direct expansion coil, preheat or reheat coil, heating coil section, filter section, mixing box, or combination mixing box filter section. In some larger units, a return fan may be added to the unit. Air handling units are configured to be either blow-through or draw-through units. Blow-through unit is when one of the coil sections is located downstream of the supply fan. Draw-through unit is when one of the coil sections is located upstream of the supply fan. Draw-through units can be further configured to be either horizontal units or vertical units.

Air handling systems are comprised of the following major components.

COOLING & HEATING COILS

Cooling & heating coils for air-conditioning applications consist of aluminum finned copper tubes. The fins increase the surface area and thus improve heat transfer. Low temperature hot water or chilled cold water is circulated through the tubes of the coils to provide the energy source for heat transfer. Control is achieved via the use of 2 or 3 port motorized control valves. The valves control the amount of water flowing into each coil, dependant upon the temperature demand. In some cases, cooling coils will use refrigerant directly rather than having chilled water as an intermediate fluid.

1. **Chilled water coils -** Fin coils generally consist of rows of round tubes or pipes that may be staggered or placed in-line with respect to the airflow. The inside surface of the tubes is usually smooth and plain, but some designs have various forms of internal fins or turbulence devices (turbulators) to improve performance. The individual tubes in a coil are usually interconnected by return bends (U-bends) to form the serpentine arrangement of multi-pass tube circuits. Chilled water coils usually have aluminum fins and copper tubes, although copper fins on copper tubes are also used.

2. **Hot water coils -** Hot water coils are basically the same as described for chilled water coils. However, the most common circuiting arrangement is often called single -row serpentine or standard circuiting. With this arrangement, all tubes in each coil row are supplied with an equal amount of water through a manifold, commonly called the coil header.

3. **Direct expansion coils -** Coils for refrigerants present more complex cooling fluid distribution problems than water coils. The fin coil that is used for evaporators in most air handling units is typically constructed from copper tubes with aluminum fins. Fin spacing is generally from 6 to 14 fins per inch (2 to 4 mm). Most manufacturers offer a wide choice of fin space and a number of tube rows, usually two to eight in a single casing. The whole tube and fin assembly is enclosed in a galvanized steel casing.

4. **Steam coils -** Coils generally consist of a steam header and a condensate header joined by finned tubes. The headers may be both at a side of the unit, with U-bends between them, or sometimes an internal steam tube is used to carry the steam to the remote end of an outer finned tube. Vertical headers may be used with horizontal finned tubes, or sometimes horizontal headers at the bottom of the unit supply vertical finned tubes. Copper and aluminum are the materials most commonly used in the fabrication of low-pressure steam coils. Low-pressure steam coils are usually designed to operate up to 150 to 200 psig (1.0 to 1.4 MPa). For pressures higher than 200 psig, tube materials, such as red brass, Admiralty, or Cupro-Nickel, are used. Tubing made of steel or various copper alloys, such as Cupro-Nickel, are used in applications where corrosive materials or chemicals might attack the coils from either inside or outside.

DEHUMIDIFIERS

A dehumidification system is one that takes the water vapor from the air. It may do this by cooling the air below its dew point or by chemical means. Two most common means for dehumidification equipment are refrigerant based and desiccant based dehumidifiers.

1. **Refrigerant Based Dehumidifiers –** Cooling coils can be used to provide dehumidification of the supply air by the reduction of air temperature below the dew-point, forming condensation, and thus reducing the moisture content of the air. The refrigeration type of dehumidifier is the most commonly used system. The refrigerated system has its limitations. The coil temperature can only be cooled to the point where the moisture on the coil does not freeze. If the refrigerated system cannot remove enough moisture from the air, other dehumidifiers, such as solid-state absorbents, are typically used.

2. **Desiccant Based Adsorbents** – Desiccant based dehumidifiers are those which have the ability to make moisture cling to their surface. The products that are used the most are silica gel, activated alumina, and molecular sieve. These desiccant materials will take the water vapor from air or gas with physical or chemical change. The water vapor that they pick up can be released by passing hot dry air across the surface of the product. These products have submicroscopic cavities that hold the particles of adsorbed water vapor.

HUMIDIFIERS

Steam humidifiers are generally used for central air handling systems. But, in order to ensure the advantages of steam humidifiers over other humidifiers, steam humidifiers must provide three performance characteristics: conditioning, control, and distribution. The humidifier must condition the steam to be completely dry and free of significant matter. It must respond immediately to control, provide precise output, and distribute steam as uniformly as possible into the air. Failure of the humidifier to provide these characteristics will result in improper humidification.

SUPPLY & EXTRACT FANS

Fans are used to provide air circulation and move other gases or vapors out of the building. There are two common forms of fans: centrifugal fans and axial fans. Centrifugal fans are able to move more air at higher pressures and with less noise than axial fans. Axial flow fans are used when higher air volumes at lower pressure resistances are required. In general centrifugal fans would be used for a ducted system, for either supply or extract air flows, where a number of grilles are served by a common ventilation system. Axial flow fans would be used for point of use ventilation systems such as localized extract fans.

All fans have three basic parts: an impeller, motor, and housing. The impeller is the part of the fan that moves the air. In order for an impeller to move air, it must rotate. This is done by power from the motor. Housings are made to fit the individual fan types. Materials used in fan construction are generally steel. They are also built of aluminum and can be made of special materials, such as stainless steel or epoxy-bound fiberglass. Fans can be coated with compounds that are especially suited to the many kinds of corrosive atmospheres in which they must work. In some cases, spark-proof construction is required.

Centrifugal Fan

A centrifugal fan is built with a wheel that is mounted on a horizontal shaft and turns in housing. Air enters near the axis of the wheel and is discharged through the housing outlet. Air may enter the fan wheel at one or at both ends of the wheel's axis. The fans that are in use for air- conditioning, heating, and ventilating systems normally do not exceed 10 inches of water (2,488 Pa) static pressure. The main feature that distinguishes one type of centrifugal fan from another is the curvature and the inclination (slope) of the fan wheel blades. The slope largely determines the operating characteristics of the fan. The three principal types of blades are the forward-curved blade, the radial blade, and the backward inclined blade.

1. **Forward Curved Fan -** The forward-curved blade fans are used primarily for the low-pressure heating, ventilating, and air-conditioning application. Domestic furnaces, low-pressure central station air handling units, and packaged air-conditioning units, such as window and rooftop air-conditioning units, use this type of fan.

2. **Radial Fan -** In the radial blade fan wheel, the tip of the blade projects straight out from the fan shaft. The radial blade fan can work at a higher pressure than either the forward-curved or the backward-inclined blade fans. However, to move the same amount of air as the other two types, the radial fan wheel requires more horsepower. Due to its low efficiency, the radial blade fan generally is not found in heating, ventilating, and air-conditioning (HVAC) applications. It is used more for material handling applications, since the wheels are of simple construction and they can be fixed in the field.

3. **Backward Curved -** In the backward-inclined blade fan wheel, the tip of the blade is inclined backwards away from the direction of the rotation of the fan wheel. This lets the backward- inclined fan move air at higher pressures than the forward-curved fan. It is more efficient (uses less horsepower) for many air volumes and pressure ranges than the forward-curved blade fan. The backward-inclined blade wheel is also built with blades that are made in an airfoil shape. This wheel is the most efficient of all types and is the quietest at high static pressure. The backward-inclined fans are generally used in medium to large air handling systems. They are normally used for medium- and high-pressure systems, although they are, at times, used in low-pressure systems.

Axial Fan

Axial fan flow is moved parallel to the shaft on which the fan wheel is mounted. Axial fans use either a direct drive or a belt drive. The three main types of axial fans are propeller fans, tubeaxial fans, and vaneaxial fans. Axial fans do not develop their static pressure by centrifugal force. The static pressure is gained from the change in velocity of the air when it passes through the fan wheel.

1. **A propeller fan** consists of a multi-blade impeller within an inlet ring or plate. Propeller fans are low-pressure, high-capacity units built either with the blades mounted on the shaft of an electric motor or a shaft for V-belt drive.

2. **A tubeaxial fan** consists of an axial flow wheel within a cylinder or tube. Tubeaxial fans may be used on low- and medium-pressure systems.

3. **Vaneaxial fans** are tubeaxial fans with guide vanes that straighten out the axial spiral airflow. Vaneaxial fans can be used in low-, medium-, and high-pressure systems.

DAMPERS

Dampers are devices used to control or restrict the airflow. They commonly consist of a series of moveable blades mounted in a frame. One of the most obvious attachments to a damper is a damper motor or actuator. These control the opening and closing of the dampers, in response to signals from the control system. They fall primarily into three types: volume, backdraft, and fire dampers.

1. **Volume Dampers -** Volume dampers are devices used to vary the volume of air that passes through an air outlet, inlet, duct, fan, air handling unit, cooling tower, or condenser unit. They may vary the volume from 0 to 100 percent of capacity. Some volume dampers can be opened and closed by hand, while others are opened and closed by a pneumatic or an electric operator. The largest use of manual controlled volume dampers in cooling systems is for air balancing.

2. **Backdraft Dampers** - Backdraft dampers are devices used to limit the airflow within a duct to one direction and to stop airflow through a duct or opening when the fan is shut off. Backdraft dampers are opened automatically by the force of the airflow on the

damper blades. They are closed automatically by a spring or weight counterbalance and by gravity. The counterbalances can be adjusted to allow the damper to pass the needed airflow. Backdraft dampers, because they are free to open and close easily, may rattle and make noise. To eliminate this, felt or vinyl strips can be placed on the damper edges, which will also help minimize the air leakage.

3. **Fire Dampers -** Fire dampers are devices used to close off individual sections of a building during a fire. Fire dampers are normally installed where a duct passes through a wall, partition, floor, or ceiling which is specifically designed to provide fire resistance. If ducts pass through barriers having a fire rating of up to and not more than one hour of fire resistance and can be assumed to present no further fire hazard, there is no need for fire dampers. If the wall, partition, ceiling, or floor is required to have a fire resistance rating for more than one hour, a fire damper is then required to properly protect the opening where the ductwork penetrates the wall. Fire damper blades are held open by a fusible link (replaceable) during normal operation of the building. If a fire occurs, the fusible link melts and the damper blades close automatically. For a cooling system to operate properly, all fire dampers must be open all the way. Broken or damaged fusible links should always be changed, and fire dampers should never be wired open. Break-away type connections should be used to connect the ductwork to the fire dampers; solid connections should never be used.

4. **Smoke Dampers -** Smoke dampers are used for either smoke containment or for smoke control. The damper is basically the same as a volume damper, except the damper is classified and listed in accordance with Underwriters Laboratories, Inc. (UL) 555S, UL Standard for Safety Smoke Dampers Fourth Edition (1999). The damper is a two-position damper, i.e., the damper is either open or closed depending upon the control requirements. The dampers are opened and closed by a pneumatic or electric operator. The damper usually has low leakage characteristics.

LOUVERS

These are the elements which allow air to be drawn in from outside or discharged to outside, without allowing rain into the building. Louvers are shaped such that rain falling or being blown onto the louver is captured and channeled back to outside. Louvers may be incorporated into the side of a building, or they can take the form of free-standing penthouse arrangements.

Each louver should be complete with a bird mesh to avoid small birds or large insects from being drawn into the ventilation system. Variations include sand filters for desert areas and double bank louvers for particularly exposed areas.

Problems encountered with louvers include louvers obstructed with leaves etc., supply and extract louvers which are located too close together allowing short-circuiting of stale air, or intake louvers located adjacent to areas in which vehicles run, allowing exhaust gases to be drawn into the building.

UNIT HEATERS

A unit heater is an assembly of a fan and motor, a heating element, and an enclosure whose function is to heat a space. Generally, unit heaters use five different types of heating media: steam, hot water, gas indirect-fired, oil indirect-fired, and electric. Propeller fan units are the most popular units used; however, sometimes centrifugal fan units are used. Unit heaters are used for spot or intermittent heating, such as large outside doors. Unit heaters are used to heat garages, factories, warehouses, stores, etc.

DUCTWORK

Ductwork is the system of ducts and ductwork accessories that are used to connect air handling units and fans with the rooms, spaces, or exhaust hoods with which they are associated. The material used for a duct system must be based on the availability of the material, expertise of the duct installer, the type of duct already installed, the location of the installed duct, and the environment it is planned to be used in. For example, a fume hood that handles corrosive fumes should be connected with a non-metallic polyvinyl chloride (PVC) or stainless steel duct. Metallic ducts are usually built from sheets of aluminum or galvanized steel. The ducts may either be built with round or rectangular cross sections. Non-metallic ducts are usually built from fiberglass duct board, except for ducts handling corrosive fumes that are constructed from a PVC material. Fiberglass duct board sheets are generally in locations where the duct will not be damaged by objects or personnel. All joints should be sealed with a special pressure-sensitive tape made for this purpose; standard duct tape should not be used. Round PVC duct systems are built from standard PVC duct and standard fittings. Fittings and ducts are connected with glue. Rectangular PVC ducts are of a special construction and should be made by people who are skilled in this work. Flexible ducts can be bought and used directly without further fabrication. They are available either insulated or non-insulated.

AIR FILTERS

Filters are installed within the incoming airstream of ventilation or air conditioning systems to remove particles of dust, smells and pollution from entering or being circulated throughout the building. The process used to remove these contaminants from the air mechanically, electrically, or by absorbing them, is called air filtration. There are many different types and different grades of filter, used to remove different particles or pollutants. Typically, flat panel or peaked fabric filters are used for simple systems, and these are capable of removing most dust particles and general dirt. The common type of filters used in HVAC applications are -

1. **Impingement Filters -** Viscous impingement filters are of the panel or roll type with a viscous (tacky) coating on the media to hold the particles to the media. The coating is called an adhesive. Viscous impingement filters are made to trap large dust particles from the airstream. Most air-conditioning systems have this type of filter. There are four types of these filters: throwaway, cleanable, automatic renewable media, and automatic self-cleaning media.

2. **Dry Media Filters -** Dry media filters, as their name suggests, do not have the tacky coating that is on viscous impingement filters. The dry media filters take out particles from the air by interception and straining. Interception means to filter out particles using the natural forces of attraction between molecules. Straining means to take out particles that are too large to pass through the openings between the fibers.

3. **HEPA Filters -** HEPA stands for High Efficiency Particulate Air. The HEPA filters work on diffusion principle to remove particulate matter and are extremely important for maintaining contamination control in clean room environments. These filter particles as small as 0.3 µm (microns) with a 99.97% minimum particle-collective efficiency. This is

remarkable considering that the outside air we breathe may contain up to 5 million suspended particles of dust, smog, and pollen in one cubic foot.

4. **Activated Carbon Filters -** A filter made of activated carbon will get rid of solid particles, as well as odor-causing gases and bacteria from the airstream. It is possible to clean and reuse the carbon filters. However, this is best done by the manufacturers, who will take out the carbon and process it to be used again.

5. **Electrostatic Filters -** The inability of standard dry or viscous type filters to take out fine dust particles from airstreams has led to the development of the electrical precipitator. The precipitation method consists of giving an electrical charge to each dust particle in the airstream by passing the air between electrodes and then collecting the dust on parallel plates as the air flows between the plates.

DIFFUSERS, REGISTERS & GRILLES

Supply, return and extract grilles are used as the final point of entry or exit from the ventilated or air conditioned space. They can be wall mounted, ceiling mounted or floor mounted, according to the air distribution required within the space.

Supply grilles must introduce the air into the space in the most effective way, without causing unpleasant draughts.

The location, type and size of the grille is important. Supply and extract grilles must also be sized such that they do not cause significant pressure loss for the air passing through them, which would adversely affect the fan power, or create noise due to higher air velocities through the grille.

Usually, the diffuser is a supply air outlet having square or circular shape. A grille is usually supply, returns or extract terminal unit and is rectangular shape. The term “register” means a grille or diffuser with integral volume control damper.

PUMPS

The most of the pumps used in the HVAC industry are single stage (one impeller) volute-type pumps that have either a single inlet or a double inlet (double suction). Double suction pumps are used in high volume applications; however either a single inlet or double inlet pump is available with similar performance characteristics and efficiencies. The centrifugal pumps for building services applications should be in accordance to Hydraulic Institute Standards and all hydronic piping is in accordance with ASME B 31.9 “Building Services Piping”. All electric motors and components shall comply with NEMA standards.

Close-Coupled, Single Stage, End-Suction Pump: The closed-coupled pump has the impeller mounted on a motor shaft extension. The pump is mounted on a horizontal motor supported by the motor foot mountings. This compact pump has a single horizontal inlet and vertical discharge. The motor and pump can not be misaligned and they take up less floor space than flexible-coupled pumps. However, to replace the special motor with an extended shaft that is used can be difficult to get after a breakdown.

Frame-Mounted, End-Suction Pump on Base Plate: The motor and pump are mounted on a common rigid base plate for horizontal mounting. Mounting requires a solid concrete pad. Base-mounted pumps shall be placed on minimum of 4" high concrete base equal or greater than 3 times total weight of pump and motor, with anchor bolts poured in place. The motor is flexible-coupled to the pump shaft. For horizontal mounting, the piping is horizontal on the suction side and vertical on the discharge side. The flexible-coupled shape allows the motor or pump to be removed without disturbing the other. However, the flexible coupling requires very careful alignment and a special guard. The flexible-coupled pump is usually less expensive than the close-coupled pump.

Double-Suction Split Case Pump: The water is introduced on each side of the impeller and the pump is flexibly connected to the motor. Typically, motor and pump are mounted on a common rigid base plate for horizontal mounting. Double-suction pumps are preferred in application over 1000 GPM because it’s very high efficiency and can be opened, inspected and serviced without disturbing the motor, impeller or the piping connections. The pump case can be split axially (parallel to the shaft) or vertically for servicing. This pump takes more floor space than end suction pumps and is more expensive.

Vertical In-Line Pumps: It is also a closed-coupled pump that has the motor mounted on the pump casing. These pumps have the suction and discharge connections arranged so they can be inserted directly into a pipe. Mounting requires adequately spaced pipe hangers and, sometimes, a vertical casing support. In the past, in-line pumps were used almost exclusively for small loads with low heads but now the widest rage of sizes is available.

Considerable space saving can be achieved using in-line pumps but extra care must be taken to assure that pipe stress are not transferred to the pump casing.

Recommendations

For HVAC applications, evaluate system conditions and select the optimum pump type and configuration based on efficiency and pump characteristics. In general following recommendations are useful:

1. Close-coupled end suction pumps or In-line circulating pumps (up to 50 GPM).

2. Base-mounted end suction pumps for circulating systems with flow rates between 50 and 500 GPM.

3. Horizontal split case, double-suction pumps for applications with flow rates exceeding 500 GPM.

4. Vertical in-line pumps shall be considered for applications with limited floor space.

5. The pumps can be arranged in primary-secondary arrangement for energy efficiency and for large installations variable frequency drive (VFD) pumps should be selected.

VALVES & PIPING

Valves installed in the air handling system are to control water flow and to isolate equipment for ease of operation and maintenance. There are several different physical types of valves.

1. **A check valve** is used to prevent fluid from going backwards; it allows flow in only one direction. If two parallel pumps are installed in a system but only one of them is running, a check valve at the pump discharge can prevent water from going back through the second pump.

2. **A gate valve** is used to isolate equipment and piping loops. A gate valve on both the suction and discharge sides of each pump would allow the pump to be removed for maintenance while the system continues to operate on the parallel pump.

3. **A globe valve** is used to throttle the liquid flow rate and provide precise control for modulating service. Most kitchen faucets are this type. Because the pressure loss

through an open globe valve is much higher than through a gate valve, globes should not be used for isolation applications. Conversely, gate valves are a poor choice for flow control situations, since they allow most of the flow to occur when they are only about 10% open.

4. **A triple-duty valve** is often specified for the pump discharge. This single valve serves the function of a check valve, isolation valve, and flow control valve. One trait that all three of these valves share is a linear action to operate (the valve opens as the valve stem rises).

5. **Butterfly valves** are most often used on larger pipes, typically 2½ to 12 inch and larger. They are often the only choice for controlling large-piping systems such as chillers, boilers, cooling towers, and thermal storage systems. These valves require the least opening torque and these control flow by rotating a disc within the valve body 90 deg from open to close. A key consideration when using butterfly valves for proportional control is their vulnerability to an extreme pressure drop in some applications. Unbalanced forces on the disc during high drops can cause oscillations, poor control, and/or damage to the linkage and actuator, even when the critical flow point is not reached. As a result, butterfly valves must be sized and selected using conservative pressure drop criteria.

6. **Ball valves** can be used for isolation, flow control or both and these provide tight close-off. In small sizes, ball valves are usually the cheapest option. These are available in two- and three-way configurations; two-way ball valves have equal percentage flow control characteristics, and the flow can be in either direction. Three-way models have linear flow control characteristics and can be used in diverting or mixing service, with some restrictions. When good but not precise control is necessary, motorized ball valves can be used instead of globe valves in many applications. These include heat exchangers; air handling unit heating/cooling water coils and steam heating coils; preheat coils; humidifiers; and unitary equipment such as reheat coils, unit heaters, convectors, radiant panels, fan-coil units, and unit ventilators. Lower cost is a major advantage of a ball valve; it might cost only half as much as a comparably sized linear globe valve or other valve type.

Globe valve or Ball valve

Globe and ball valve are most frequently used valves for modulating service. While motorized ball valves can be used in most common non-central plant HVAC control applications, they cannot always replace globe valves. Globe valves have strengths that make them popular for use with HVAC controls. Here are other factors to consider:

1. Globe valves should be used when precision control is a higher priority over broad range of conditions.

2. Globe valves are available in may pipe sizes, typically from ½ to 6 inches.

3. In a modulating control loop, 1/2 in. ball valves with very small reduced ports are most suitable for two-position control due to the lack of a "throttling" ability of the small port.

4. If noise is a primary concern, a globe valve may be a better choice.

5. Globe valves can handle a wide range of Cv capacities (from <1 to 400), flow characteristics, temperature, and pressure requirements. For low-flow modulating control at Cv <1, globe or zone valves are suggested.

CONTROL VALVES

Control valves are used to maintain space temperature conditions by altering the flow of steam, water, gas, and other fluids within an air-conditioning system. Valves must be properly sized and selected for the particular application. Valves can be two-position or modulating 3-port configuration.

Two-way valves throttle flow while three- way divert or mix flow. Two-way valves have two ports and are used to control the flow in variable flow systems. Three way valves are commonly used in hydronic heating systems, although the name is really a misnomer. While these valves do have three ports, the fluid enters one port and exits either the second or third port, depending on the valve setting. Three-way valves are also available with two entry ports and a common discharge port. The type of three-way valve selected will determine its location in the system.

Mixing Valves- A three-port valve with two inlet flows and one common outlet flow is defined as a mixing valve, and so provides a variable temperature outlet at a constant flow rate. A three-port motorized valve can be used to MIX, in varying proportions, two flows of different temperatures while maintaining a constant rate of flow in the common outlet port. A Mixing Valve is used normally for radiator circuits.

Diverting Valves- A three-port valve may also be used to DIVERT a common flow in varying proportions. The valve will have one inlet and two outlets and provides a constant temperature and variable flow rate. A diverting valve is used normally for circuits with convective heat transfer such as; heat exchangers, primary coil in indirect cylinder, heater battery, cooling coil. Diverting valves in bypass applications are placed upstream of the coil. Diverting applications are commonly used in constant flow systems where full flow across the coil is not required because of partial load system conditions.

2-way Valve or 3-way Valve

2-way valve is best when applied with variable speed pumps. Rather variable-pumping systems should only use 2-way valve to reap the energy saving benefits.

The constant volume systems may employ 2-way or 3-way valve. While pumping costs will decrease to small amount with 2-way valve, other problems occur. The 2-way valves with constant volume system may some time lead to balancing problems in large network and may lead to water scarcity at some terminal locations. The pumps must incorporate the minimum recalculating system should the 2-way valve/s close to 100% close position. The systems incorporating 3-way valve ensure continuous circulation.

All of these valves can be operated manually, or fitted with an actuator and controlled automatically. When on/off control is required, a solenoid valve is specified. When the flow must be allowed to vary from closed to full capacity, a modulating valve is selected. Many types of systems require the installation of a bypass valve around a piece of equipment to control the flow rate through the equipment.

HEAT RECOVERY DEVICES

Heat recovery devices are used to transfer heat between the intake fresh air and the exhaust air. This is useful in situations where a large amount of air has to be expelled, particularly when it is cold. In an office building situation, the fresh air quantities are normally quite small and this can make heat recovery fairly unattractive. It is very important to make sure that you have addressed all the possible savings from improved fresh air and recirculation control before considering heat recovery.

Options for heat recovery include:

1. **Run around coil -** In this arrangement, water is circulated between coils in the two ducts. A run-around coil system is a simple piping loop with an upstream precooling coil and a downstream reheating coil that sandwiches the main cooling coil. The circulating fluid is pumped to transfer heat from the warm mixed air to the off coil cold supply air. The run-around system reduces the cooling load on the main cooling coil; reheat is provided by the heat picked up by the circulating fluid in precooling coil instead of by an external source of expensive energy.

2. **Plate heat exchanger -** In this arrangement, a heat exchanger is situated to allow direct transfer of heat from one flow to the other. Heat is transferred from outdoor air coming into the air conditioner to the cold air leaving it. Again, the goal is to boost the portion of air conditioning capacity used for removal of latent heat by decreasing the need to remove sensible heat. To a limited extent, the exchanger itself will remove latent heat by condensing moisture on the entering air side. Outdoor air is introduced into one side of the heat exchanger, and is partially cooled. It then flows over the cold refrigeration coil for moisture removal and additional sensible cooling. The cold, saturated air then passes through the other side of the exchanger for warming before being introduced to the air handler or the space.

3. **Heat Pipe System -** In its simplest form, a heat pipe is a metal tube sealed at both ends, evacuated and charged with a vaporizable liquid (refrigerant). The liquid refrigerant at the bottom end readily turns to gas when that end of the pipe is warmed, and floats to the top end of the pipe. If that end is in a cooler environment, the gas condenses, releasing heat. The liquid then flows back to the bottom where the cycle begins again. The net result is heat transfer from bottom to top, without a compressor. Capillary action is sometimes used to help move the liquid, allowing for greater flexibility in configuration.

4. **Thermal Wheel -** In this arrangement, a rotating wheel is used to transfer heat from one air stream to the other. Energy recovery wheel can be incorporated into a ventilation system to transfer both the sensible and latent energy between outdoor and exhaust air

streams. The core of an enthalpy wheel can be made from a variety of materials such as paper, metal or plastic, which is coated with a desiccant. This coating enables the wheel to transfer both sensible and latent energy between air streams. The enthalpy wheel provides first stage cooling or heating. For instance, in the summer the wheel removes moisture from incoming air and pre-cools it to lighten the load on the evaporator coil. Likewise, in colder temperatures (winter application) the wheel will inject moisture into supply air and pre-warm it to reduce the load on the reheat coil. In the summer and winter modes, the wheel saves energy. There are three general types of wheels being used today. They are sensible, enthalpy and regenerative.

- **Sensible wheel -** This wheel is not coated with a desiccant and therefore transfers only sensible energy. The wheel can be constructed of almost any material (paper, metal or plastic) and transfers energy between two air streams as the mass of the material gains or loses heat to the opposite air stream. The wheel rotates at a speed of 25 to 50 revolutions per minute.

- **Enthalpy wheel -** It is similar to the sensible wheel except that a desiccant media is added to the wheel's surface. As the wheel rotates, it now can transfer sensible energy and humidity. This wheel also rotates at 25 to 50 revolutions per minute.

- **Regeneration wheel -** This wheel is used when low dew-point conditions (<45°F) are required, such as industrial applications. It achieves low dew-points by slowing the wheel to a speed of between 0.25 and 1 revolution per minute and by using an air stream heated to 250°F or more to drive off moisture and regenerate the wheel. This heated air stream is typically focused on only 1/4 of the wheel's area thereby allowing 3/4 of the area to be available for the process side.

SECTION-3 HVAC CONTROL SYSTEM EQUIPMENT AND CONTROL LOOPS

The design of HVAC control systems is implemented by defining the operating modes of the HVAC equipment, defining the control loops required, and selecting the control system equipment to be used in the loop. The process of selecting the control system equipment includes calculations by the HVAC engineer to specify the flow capacity of control devices, the physical size of control devices and the electric service required. This section describes the operating modes, process variables, control modes, control system devices and their features, control system equipment applications, and inter-connection of control devices.

ELEMENTS OF A CONTROL SYSTEM

HVAC control system, from the simplest room thermostat to the most complicated computerized control, has four basic elements: sensor, controller, controlled device and source of energy.

1. Sensor measures actual value of controlled variable such as temperature, humidity or flow and provides information to the controller.

2. Controller receives input from sensor, processes the input and then produces intelligent output signal for controlled device.

3. Controlled device acts to modify controlled variable as directed by controller.

4. Source of energy is needed to power the control system. Control systems use either a pneumatic or electric power supply.

CONTROL SYSTEM OPERATING MODES

Control systems start and stop the HVAC system equipment according to a time schedule, and at specific outside air temperatures and specific indoor temperatures. In addition, the control systems may be required to operate the HVAC systems in the following modes of operation:

1. Occupied mode is initiated automatically to allow HVAC systems to start in sufficient time to bring the space to the proper temperatures at the start of occupancy.

2. Ventilation delay mode is initiated automatically to prevent the use of outside air when the unit is started prior to occupancy, to cool down or warm up the area served.

3. Unoccupied mode is initiated automatically to prevent unnecessary operation of HVAC-system equipment during periods of non-occupancy except for special purposes such as operation to maintain minimum space temperatures for freeze protection.

4. Heating or cooling modes are initiated manually to provide either heating or cooling media to HVAC equipment.

CONTROL SYSTEM PROCESS VARIABLES

While the HVAC systems are in operation, the process variables commonly sensed and controlled by HVAC control systems are:

1. Temperature

2. Relative humidity

3. Static pressure of air

4. Differential pressure of air

5. Air flow rate

Some common devices that sense the process variables and/or control the output equipment are:

1. **Thermostat -** A thermostat or other temperature control device may move a damper that directs the path of an airstream. It may also change the temperature of an airstream by directing its flow through a coil, or it can control the volume of air flowing in a duct system. Through the action of a valve, it also may change the temperature of an airstream.

2. **Humidistat –** A humidistat or humidity controller can be switched between humidify and dehumidify for removing unwanted moisture accumulating within an enclosed area or to add needed moisture to the air by means of humidification.

In the dehumidifying mode high humidity conditions will provide signal to the 2-way or 3-way control valve to regulate the chilled water supply to the cooling coil. The humidistat signals normally take precedence over the temperature signals. In the humidifying mode this control operates fogging or other humidifying equipment by activating switches, motors, valves or pumps. Humidifiers can be precisely controlled according to atmospheric humidity to create ideal conditions for any environment. Sensor allows a 5% (differential) comfort zone between "on" and "off" function

3. **Pressure Controls -** Pressure controls operate to control the pressure of air in a room, or the duct system through the action of dampers. The damper being controlled is in the inlet duct to the supply fan, but it could equally well be an inlet vane damper or an outlet damper.

   The pressure controller could also operate a blower drive speed control. In this control, the pitch of motor and fan pulleys is varied by a device in response to a signal from the pneumatic static pressure controller.

4. **Fan Controller -** Control of fan speed to control pressure and airflow volume results in less noise than when inlet vanes or outlet dampers are used. In certain installations, the use of speed control also cuts down the horsepower that is needed by the fan or blower drive motor.

5. **Airflow Switches -** Airflow switches are mounted on the side of a duct with the blade inserted into the duct. The blade of the switch will move according to airflow in the duct, and it will make electrical contacts when air flows and break the contacts when airflow stops. The sensitivity of the switch to the airflow may be adjusted. Another control for the same purpose as the ones described above is the differential pressure switch that senses velocity pressure in the duct. This controller also has an adjustable pressure range.

MODULATING CONTROL

The amount of heat delivered to a space (or removed from a space) from certain types of HVAC equipment is regulated by varying the heat exchanger capacity from zero to one hundred percent in response to the variation of a continuous, gradual input signal. This is called modulating control. Heat exchanger control valves, mixing dampers, fan inlet vanes, variable speed drives, and humidifier valves are examples of HVAC equipment that are controlled by modulating control.

TWO – POSITION CONTROL

The amount of heat delivered to a space from certain types of HVAC equipment is controlled by turning the equipment on and by shutting the equipment off. This type of control is also called on-off control. Examples of 2-position control are the starting and stopping of the fans of unit heaters and fan coil units by room thermostats to maintain space temperature, and the opening and closing of shutoff dampers when fans are started and stopped.

CONTROL SYSTEM EQUIPMENT

Control Valves

Control valves are used to regulate the flow of fluids in piping systems by compressing and releasing a valve spring to move a valve closure disk or plug toward or away from the closure seat of a flow port. The valves are used both in modulating and in 2-position control applications.

1. Examples of the use of modulating control valves are:
   - Heating and cooling coil control valves
   - Converter steam control valves
   - Humidifier control valves
   - Perimeter radiation system zone valves
2. Examples of the use of 2-position control valves are:
   - Dual-temperature water system changeover valves
   - Shutoff valves used in fan coil unit coils

3. Control valves are classified according to their flow regulating body patterns. A 2-way valve restricts fluid flow in one direction, because it has one inlet and one outlet; a 3-way valve restricts flow in two directions. A 3-way (mixing) valve has two inlets and one outlet, and a 3-way bypass (diverting) valve has one inlet and two outlets.

4. In the flow mixing application, the 3-way valve is used to mix heated primary flow, from a boiler or a converter, with system return flow to produce system secondary supply, for the purpose of controlling temperature. When used on the return line from a coil, one of the 3-way valve's inlets is from the coil, and the other inlet is from the bypass around the coil. 3-way mixing valves may be used for controlling the following types of HVAC equipment to prevent deadheading of pumps (in lieu of the 2-way valves):

    - Cooling coils served by <u>variable volume pumping systems </u>shall have 2-way valves.

    - Cooling coils served by <u>constant volume pumping systems shall have 3-way valves</u> to prevent deadheading of pumps. 3- way valves are also recommended as diverting valves around boilers or cooling towers

Control Valve Characteristics

The flow regulating characteristic of a valve is generally determined by the shape of a disk or plug that passes through the flow port. Typical flow regulating characteristics used for control systems are:

- Linear flow, in which the percent of valve travel, equals the percent of maximum flow rate through the valve – (generally applicable to 3-way mixing valves).

- Equal-percentage flow in which equal increments in the percentage of valve travel produce an equal-percentage change in flow rate from the previous flow rate, when a constant pressure drop is maintained – (generally applicable to 2-way valves).

The equal-percentage flow characteristic matches the non-linear heat exchange characteristics of the HVAC equipment coils with a change in fluid flow that tends to linearize the heat exchange output of the coil with a linear signal to the control valve. The linear-flow characteristic is more suitable for mixing applications and for humidification.

Control Dampers

Dampers are used to regulate the flow of air in ductwork in both modulating and 2-position control applications.

1. Examples of the use of modulating dampers are:

    - Air plenum temperature control by mixing outside air and return air.
    - Space temperature control by mixing warm air and cool air.
    - Space temperature control by varying the flow of cool air.

2. Examples of the use of 2-position dampers are:

    - Closing outside air dampers or building exhaust dampers when fans are stopped.
    - Isolating sections of ductwork for smoke control purposes.

3. Dampers are classified by the action of their blades, which connect to a common shaft that is rotated to open or to close the damper.

    - <u>Opposed blade</u> dampers provide better flow characteristic in throttling applications. A throttling application is one where the damper is installed in series with the path of flow and the damper is used to add pressure drop to reduce air flow.
    - <u>Parallel blade dampers</u> are used to provide better flow characteristics in mixing applications. A mixing application is one where more than one flow path exists in parallel. Usually, two or more dampers are installed in parallel to each other and the dampers divert flow rather than increase total system pressure drop.

4. When a control system application requires that a damper be open prior to the start of a fan, an adjustable switch is connected to the damper; this device is called an end switch or limit switch. The end switch operates a set of contacts in the fan starter control circuit when the damper is fully open, to allow the fan to start; the end switch opens the circuit to prevent the fan from continuing to operate if the damper begins to close.

Actuators

Actuators are used to operate valves and dampers. Pneumatic actuators are powered by air pressure, and are controlled directly by a pneumatic control signal and indirectly by an electric or electronic signal. An electro-pneumatic device converts an electric or electronic signal to a pneumatic signal to stroke the actuator. Electric and electronic actuators are electrically powered and are controlled directly from an electric or electronic signal to stroke the actuator. While all pneumatic actuators have a spring-return feature, some electric/ electronic actuators are not equipped with a spring to move the valve or damper to a fail-safe position upon loss of power or control signal.

1. Modulating control of actuators requires either the use of a 4 to 20 milliampere control signal directly to an electronic actuator or the conversion of the signal to a pneumatic control signal of 21 to 103 kPa (3 to 15 psig). The pneumatic signal can be directly or inversely proportional to the electronic signal.

2. Two-position control of electric actuators requires the closing and opening of a contact to operate an electric actuator. Two-position control of pneumatic actuators requires an electric/pneumatic device to pass 140 kPa (20 psig) main air to the actuator, or to exhaust air from the actuator.

3. Sequencing occurs when actuators are modulated from a common signal by using a portion of the 4 to 20 milliampere signal or the converted 21 to 103 kPa (3 to 15 psig) signal. The actuator stroke is adjusted to move its connected valve or damper from fully closed to fully open over the assigned portion of the common control signal. Dead bands between the movement of valves and dampers are achieved by assigning a portion of the common control signal as a deadband. Each actuator is adjusted so that its full stroke occurs on either side of the deadband limits outside of the deadband.

4. The actuators that control valves and dampers are sequenced when HVAC applications require that the process variables be sensed at a common location and controlled from a common modulating signal. The objective of sequencing is to avoid energy waste by preventing the following opposing processes from acting simultaneously:

    - Heating and cooling
    - Humidification and dehumidification

Simultaneous heating and cooling can occur when pneumatic actuators are used, even though the spring operating ranges are selected without an overlap. Because of this phenomenon, sequencing applications for HVAC systems must have positive positioners on pneumatic valves and damper actuators, to maintain deadbands between actuator operating ranges.

- All modulating control applications of pneumatic actuators require that the actuator be equipped with a positive positioner (PP).
- Non-modulating (two-position) control applications where pneumatic actuators are used do not require positive positioners.

Current –to – Pneumatic Transducers: The modulating device for converting a current control signal to a pneumatic control signal is a current-to-pneumatic transducer (IP). A 140 kPa (20 psig) main air supply to the IP is the source that develops a 21 - 103 kPa (3 - 15 psig) output signal in a scaled relationship to a 4 - 20 milliampere input signal.

Solenoid Operated Pneumatic Valves: The 2-position device for converting an electric contact closure signal to a pneumatic signal is the solenoid operated pneumatic valve (EP). The EP is a 3-way valve that connects the normally closed and common ports when the solenoid coil is energized and connects the normally open and common ports when the solenoid coil is de-energized.

Choice between Pneumatic and Electric Actuators

All terminal unit control systems shall have electric or electronic actuators. For all other control system applications, the HVAC designer must make an estimate of the total cost of actuators required for all control systems in the project. The designer will take into account the cost of multiple actuators on large dampers and the cost of larger actuators required for higher torques to operate large valves.

The total installed cost estimate of pneumatic actuators will include:

- The actuators
- Current to Pneumatic Transducers (IPs)

- Tubing
- Local indicators
- The cost of the compressed air system

The total installed cost estimate of electric actuators will include consideration of:

- The actuators
- Wiring
- Loop driving circuits
- Power transformers (24 VAC)

I/O Points

Input points provide information to the controls system, e.g. temperatures, pressures etc. Output points are the control signals to equipment. Each input or output (termed an I/O point) provides an electronic interface between the central controller (also termed energy management system EMS) and local controllers. Inputs and outputs can be either:

1. Digital, i.e. ON or OFF, such as the signal to turn a system on and off at the beginning and end of a day. For example a digital sensor can be used to provide the controller with a discreet signal (open or closed contacts) such as a pump that is on or off.
2. Analogue, i.e. continuously variable over a range, such as a temperature signal. Analog sensors are used to monitor continuously changing conditions. The analog sensor provides the controller with a varying signal such as 0 to 10V.

Special modules are required to provide the I/O points and each system has a limitation on the total number of points that can be connected. Naturally the more points a system is capable of monitoring or controlling, the higher the system costs. Analogue points are also usually more expensive than digital points.

The I/O is arranged in blocks to suit a computer addressing method so they are usually purchased as standard modules containing 4, 8, 16 or 32 points. Separate modules are usually required for the dedicated type of I/O i.e. input, output, analogue, digital, range or voltage level etc. When new points are required a complete module may have to be installed.

Single Loop Digital Controller

Single loop digital controllers are used essentially for all HVAC systems other than simple unitary systems and terminal units that are controlled directly from room or zone thermostats. The single loop local controllers are used for the following applications:

- As a controller for maintaining temperature, relative humidity, static pressure, and/or airflow setpoints.
- As an economizer mode switchover controller that determines whether outside air is suitable for cooling.
- As an outside air temperature controller for scheduling hydronic heating supply temperature and for starting and stopping pumps.

In all applications where it is used, the controller will be mounted in a HVAC control panel.

1. The single-loop digital controller typically has the following inputs and outputs:

   - A process variable analog input (PV).
   - A remote setpoint analog input for control point adjustment (CPA).
   - An analog output (OUT).
   - A process variable actuated contact closure output (PV contact).
   - A contact output that responds to the difference between PV and CPA analog inputs.
   - An analog output which is identical to the process variable input.

2. The output control modes that can be used in combination are:

- Proportional mode, which varies the output proportionally to the error between the PV input value and the controller setpoint.
- Integral mode, which modifies the output signal as a time related function of the error between the PV input value and the controller setpoint.
- Derivative mode, which modifies the output signal as function of the rate of change of the error between the PV input value and the controller setpoint.

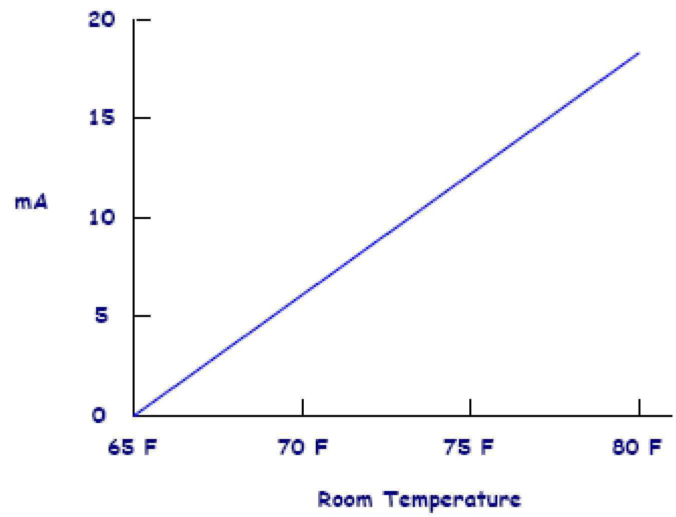
There are control functions required with less frequency in HVAC control applications than those included in the prescribed version of the single-loop controller. Control devices to perform specific control functions not available in the single-loop digital controller are called function modules.

Controller Action

All controllers, from pneumatic to electronic, have an action. They are either ‘Direct Acting’ or ‘Reverse Acting’.

Direct Action

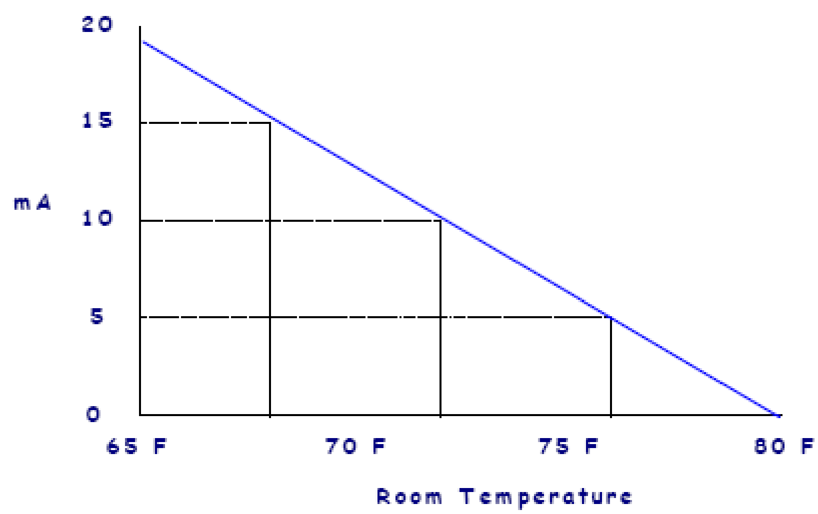
Direct Action means that the controller's output increases as the sensor's input increases. For example, as room temperature (the variable) changes from 70°F to 71°F, the controller changes its output from 10 to 12 mA. Shown below, as the sensor reads an increasing input (temperature), the controller responds by increasing its output (pressure) to the valve, closing the normally open valve and reducing the hot water flow:



DIRECT ACTION CONTROLLER

Reverse Action

Reverse Action means that as the variable (for example, temperature) increases, the controller's output decreases. For example, as room temperature rises from 70 to 71°F, the controller output decreases from 8.1 to 7.3 mA. In the example below, as the sensor reads an increasing temperature, the controller responds by decreasing its output (pressure) to the valve, closing the normally closed valve and reducing the amount of heating. This relationship is displayed on a graph as follows:



REVERSE ACTION CONTROLLER

The action of the controller must match the proper HVAC application. Normally open heating valves always use direct acting controllers. If a reverse acting controller were to be placed on a normally open heating valve, the heating valve would open as temperature rises. A reverse acting controller never properly controls a normally open heating valve.

Identifying Reverse or Direct Action

Identification of a controller action for different applications could be determined from the table below.

To use the table, follow the steps below:

1. First pick the correct application: for temperature, select heating or cooling; for humidity, select humidification or dehumidification; for pressure, select whether the sensor is downstream or upstream from the controlled device.

2. The second step is to determine how the application fails. Example, does the heat fail to full ON or full OFF?

3. Third, once these questions are answered, follow the column down and follow the row across to the intersection to find the correct action for that application.

Controller Action	Heating, Humidification, Pressure (sensed downstream from controlled device)	Cooling, Dehumidification, Pressure (sensed upstream from controlled device)
System Fails to ON Normally open ports, valves or dampers Normally closed electric contacts	Direct Action	Reverse Action
System Fails to OFF Normally closed ports, valves or dampers Normally open electric contacts	Reverse Action	Direct Action

How to use the Table; follow the examples & steps below:

Example #1: Chilled water air-conditioning cooling application requires controlling a normally open chilled water valve.

1. Under normal operation, the chilled water valve is open and as the temperature decreases, the controller signals to close the normally open chilled water valve.

2. Since the valve is normally open, the "Fails to ON" row is used.

3. The column and the row intersect at Reverse Action. As the temperature increases, the signal drops, allowing the chilled water valve to go to its normal open position. As the temperature decreases, the signal increases, closing the normally open chilled water valve.

Example # 2: A return air humidity sensor modulates a normally closed chilled water valve for dehumidification. What action is needed for the controller?

Direct Action

Example #3: A static pressure sensor (located on discharge side) modulates normally closed inlet vane dampers to maintain 2.0" w.c. (500 Pa). What action is needed for the controller?

Reverse Action

Example #4: A room sensor cycles DX cooling to maintain a room temperature of 75°F . The DX Cooling has normally open electrical contacts. What action is needed for the controller?

Direct Action

Example #5: The mixed air sensor (located on discharge side) modulates the normally closed outside air dampers and the normally open return air dampers to maintain a temperature of 55°F. What action is needed for the controller?

Direct Action

Transmitters

Variables such as temperature, pressure, and relative humidity are sensed by means of elements that are connected to the control loops via transmitters. The transmitters are 2-wire, loop-powered (i.e., powered by the control panel power supply) devices that connect in a series circuit with the controller input.

The output signals of the transmitter is the standard 4 to 20 milliampere DC signal, which is factory calibrated for zero point and span relative to the input resistance value of the sensing element. The impedance limitation of the circuit in which the transmitter can function is product specific. A typical value is 700 ohms at 24 volts dc.

Relays

Relays are used for control system interlocking functions and will be located in the system's HVAC control panel. All relays, including time-delay relays, shall be 2-pole, double throw devices.

Time Clocks

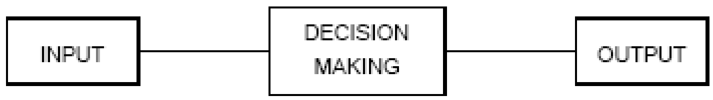
A time clock is used to control the timing of the modes of operation of an HVAC control system when the control system is not interfaced with centralized energy management control system (EMCS) or building management systems (BMS).

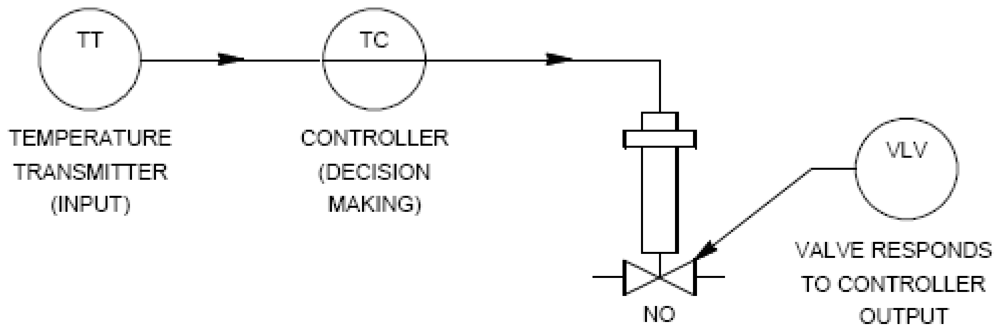
The modes of operation are occupied, unoccupied and ventilation delay. When used to time the modes of operation of air handling systems, one contact of the clock will be used for occupied and unoccupied timing; the second contact will be used for ventilation delay mode timing. For other applications, the contacts may be used as convenient to the design.

CONTROL LOOPS

A control loop performs three distinct functions: sensing of a variable as the input to a controller; decision making or control based on the value of the input; and output or actuation as a result of control.

Figure below illustrates a simple control loop.





The input signal is a continuous analog of the process, and the controller either continuously sees the input or continually scans it. The controller changes its output as required by changes in its input.

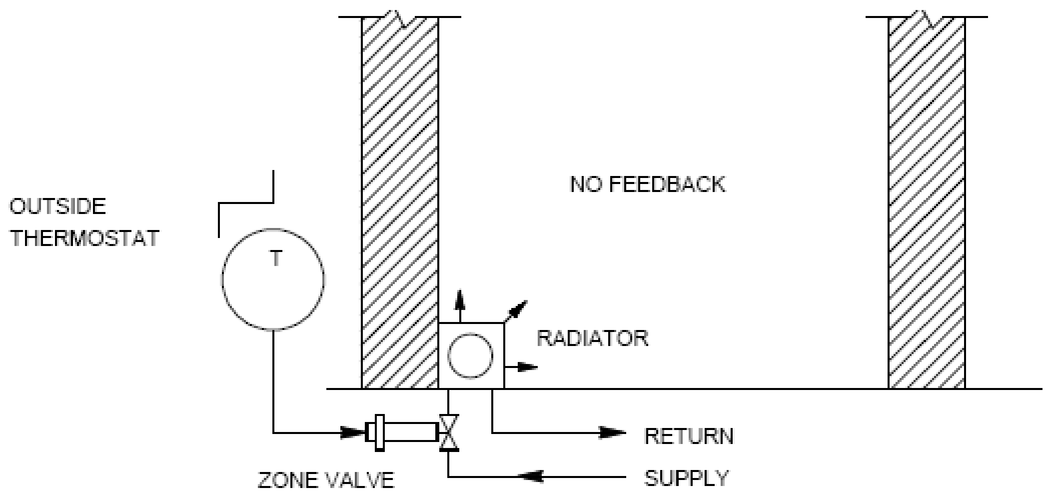
Open Control Loops

When a control loop senses a variable, makes a control decision, and sends an output signal to a control device without receiving input information related to the results of its control action, the control loop is said to be an open loop. It has no feedback i.e. there is no way to monitor if the control system is working effectively. Open loop control is also called feed forward control.

Some open-loop control applications used in HVAC control are:

- Operation of pumps above or below a certain outside temperature.
- Automatic stopping of HVAC systems based on outside air temperature.
- Scheduling of hydronic heating supply temperatures based on outside temperature.
- Timing and time-delay operations.

Figure below illustrates an open control loop.



Open control loop

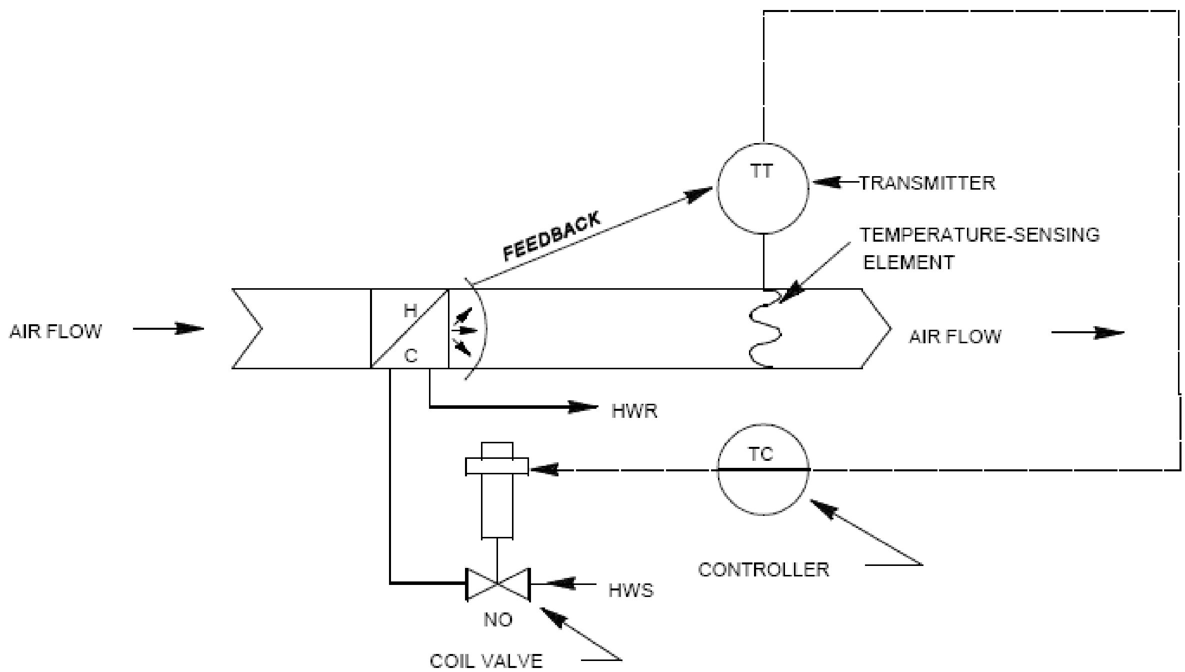
Closed Control Loops

When the controller changes its output decision based on updated input information, the control loop is said to be a closed loop. In the closed system, controller responds to error in controlled variable. A comparison of the sensed parameters is made with respect to the set parameters and accordingly the corresponding signals shall be generated. Closed loop control is also called feedback control.

Most of the control loops used in HVAC control are closed loops. Control of coil air discharge temperatures is an example. The transmitter, connected to a temperature sensing element in the air stream passing through the coil, signals the temperature controller; the controller makes a decision as to whether to open or close the valve that allows water to flow through the coil; and an actuator operates the valve. The feedback in this example is the continuous input to the controller of a changing temperature signal from the coil air discharge temperature sensor and transmitter.

The transmitter continuously updates the controller on temperature information from the sensor, and the controller modifies its output to control the valve. See figure 2-9 for an example of a closed control loop.

Figure below illustrates a closed control loop.



Closed control loop

TYPICAL CONTROL MODES

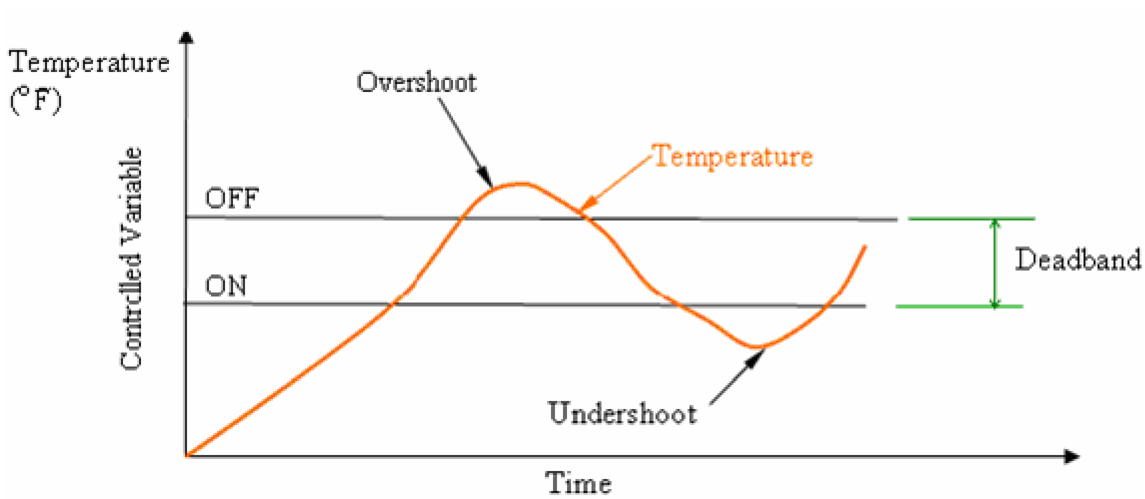
Two-position control

Some HVAC equipment can be turned on and off as a method of temperature control. This type of HVAC equipment is not applied where temperature control between close limits is required.

When a thermostat or other control device cycles equipment to maintain its setpoint the control mode is called two-position control. A thermostat used for two-position control opens and closes contacts for control rather than providing a modulating output signal. The contacts either open or close when the temperature is at the thermostat setpoint. The state of the contact reverses when the temperature changes in the proper direction. Such thermostat contacts usually either open on a temperature rise (in a heating application), or close on a temperature rise (in a cooling application). The temperature at which this happens depends on the switch temperature differential (hysteresis).

An example of two-position control is unit heater control, in which a space thermostat turns on a unit heater when the space temperature drops to 65° and turns it off when the space temperature rises to 67°F. The thermostat is said to have a differential of 2°F and a setpoint of 65°F. This type of control can result in a slight undershoot below the lower end of the differential, and a slight overshoot above the higher end of the differential.

The output from the device is either on or off, with no middle state. A diagram of two-position control as it relates to time and temperature is shown below.



Two Position On-Off Control Action

The response curve is always cycling between two limits 'on' and 'off' and overshoot and undershoot occurs in practice because of variations in heat load due to the thermal inertia of buildings and plant.

The difference between the temperatures at which the controller turns 'on' or 'off' is called the 'Differential' or 'Deadband'. If the deadband is large then control becomes ineffective.

Control or mechanical differential is difference between "on" and "off" values of controlled variable.

Operating or thermal differential is difference between extreme values (overshoot and undershoot) of controlled variable, which in above example is the swing in the actual room temperature.

The operational differential is wider than the control differential (Operating differential > Control differential) because the actual room temperature always lags behind the equipment turning 'on' or 'off'.

Advantages

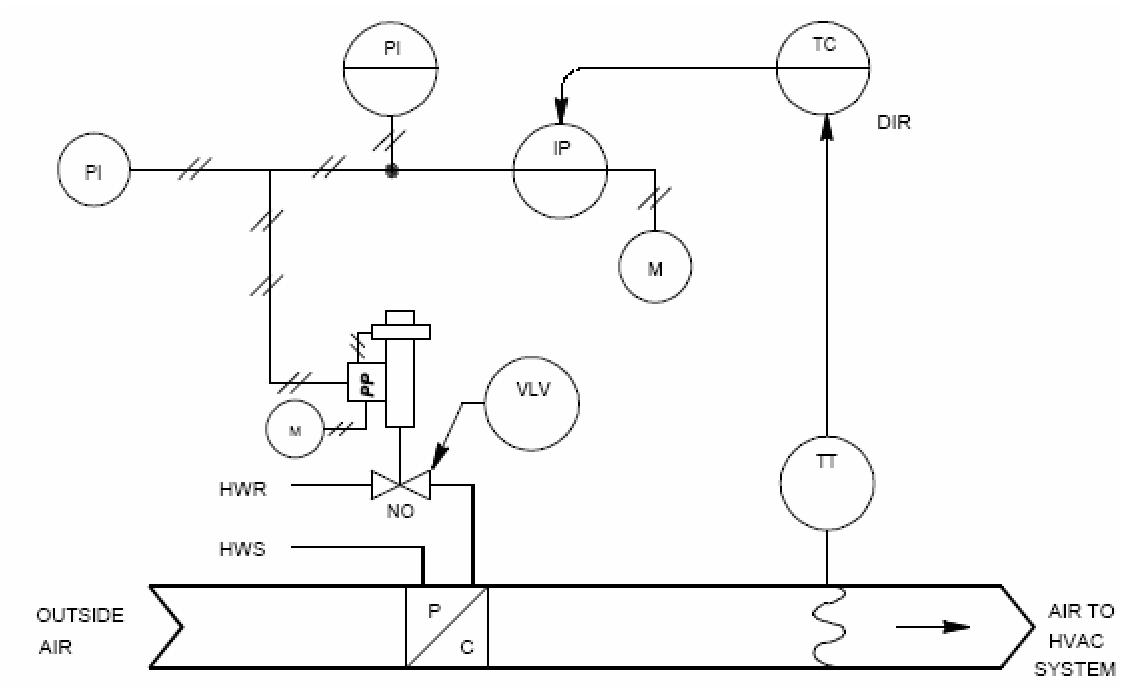
This is the simplest type of control action (and controller) and being relatively inexpensive is used in many applications where it is acceptable to maintain the controlled variable within a range. It is more suitable for plant having a slow reaction rate and high capacity on the demand side e.g. 2-port or 3- port valve on the pipework to a hot water boiler.

Drawbacks

The drawback of this type of control is that it is relatively imprecise and inaccurate. It is usually used where a precise control is not necessary, where the mass of the system is so great that temperatures change extremely slowly. On/off control must be properly matched to the system dynamics.

Modulating Control

A simple control loop is shown in figure below as it would be applied to heat outside air for ventilation using a pneumatic valve actuator rather than an electric or electronic valve actuator.



Simple control loop applied to outside air heating

The controller operates an IP in response to the signal of the temperature sensing element in the air duct, downstream of the coil, via a transmitter. The IP pneumatic output signal modulates the positioner on the pneumatic valve actuator. The positive-positioner output throttles main air to the actuator, which moves the valve stem. This example is used to explain two modes of modulating control that are applicable to the control of valves, dampers, inlet vanes, and other devices. The modes applicable to most HVAC control applications are:

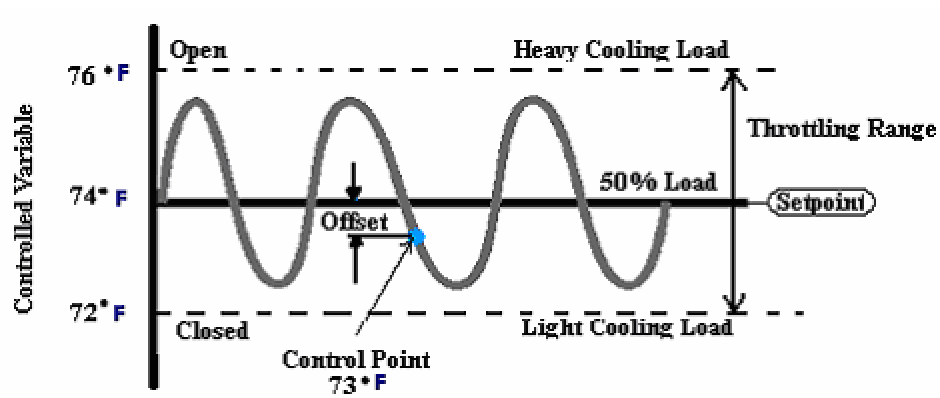
- Proportional mode (P)
- Proportional plus integral mode (PI)

PROPORTIONAL CONTROL MODE (P)

With this form of control the valve or damper is positioned in intermediate positions in proportion to the response to slight changes in the controlled condition. Therefore the controlled device does not run through its complete stroke as in the case of two-position control. Also the controlled device does not continue to move until it reaches a limit as in floating control. Instead, with this form of control, the controlled device immediately assumes a position in proportion to the system requirement. A linear relationship exists between the input and the output. A setpoint, throttling range and action typically define this relationship. This is finer control system than two-position system, and is designed to eliminate the cycling associated with on-off control. It is typical in large HVAC systems.

Example: A modulating valve controls the amount of chilled water entering a coil so that cool supply air is just sufficient to match the load at a desired setpoint. If the temperature is further from the setpoint, the on- and off-times vary in proportion to the temperature difference. If the temperature is below setpoint, the output will be "on longer"; if the temperature is too high, the output will be "off longer."

Proportional control maintains a setpoint with variations above and below that temperature. A graph of proportional control used with room cooling is shown below.



Proportional Control Action

The figure, we can see that even though the setpoint is 74°F, the temperature doesn't stay constant. It rises and falls, wasting energy and mechanical cooling, and causing uncomfortable temperature swings.

Setpoint: Setpoint is the desired condition of a variable that is to be maintained, such as temperature. The setpoint is an instruction to the control loop and corresponds to a specified value of the controlled device, usually half travel. In figure above the setpoint is 74°F. A room that needs relative humidity to be at 50% RH, or an air handler duct pressure that is to be 2.0 inches of water column (500 Pa) is examples of setpoints.

Control Point: The value of the measured variable at any given moment is called the control point. The control point is the actual temperature being sensed. The control point (temperature) may not be on the setpoint, but instead may be above or below it. Systems operate to maintain the setpoint, plus or minus some acceptable limits called differential (two-position or on/off control) or throttling range (proportional control). Simply stated, setpoint is what you want, while control point is what you get. In the example below, the setpoint is 74°F (23°C), and the control point is at 73°F and varying.

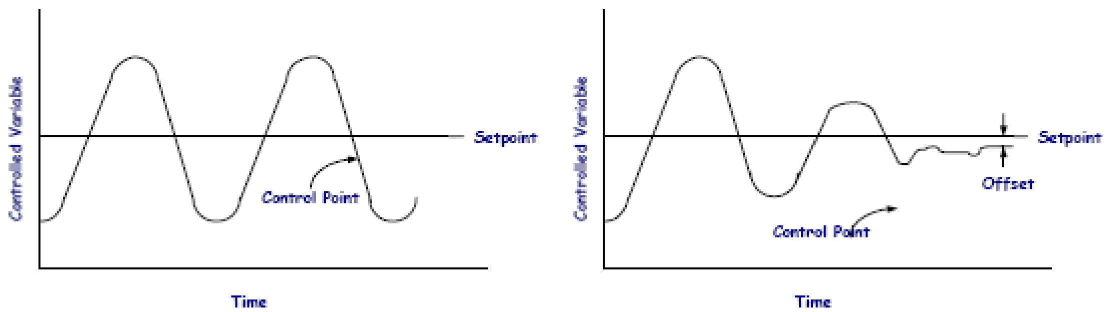
Offset: Offset is the amount away from setpoint, or the difference between the setpoint and the control point. In the example above, the offset is approximately 1°F.

Throttling Range: Throttling range (TR) is the change in the measured variable (i.e. temperature) that causes the controlled device to travel from one end of its stroke to the other. In the example above, it takes a TR of 4°F to cause the actuator to travel from the completely open position to the completely closed position. Typical throttling ranges are 8° to 10°F for mechanical controls such as mixed air control and the control of hot water supply. In contrast, room controls must be much tighter, with a throttling range between 2° and 4°F. Throttling range is sometimes referred to as insensitivity.

Smaller TR may cause stability problems. If the throttling range becomes too narrow, it causes the actuator to go into a mode called hunting. In this mode, the actuator continually searches (or hunting) for the proper controlled position full open, then full closed; then full open, then full closed, etc. A control system that is hunting is not in control. It may be possible to eliminate hunting by increase the throttling range so that the controller is less sensitive.

Stability is tendency of a system to find a steady control point after an upset.

Instability is tendency for oscillations to grow.



UNSTABLE SYSTEM UNDER PROPORTIONAL CONTROL A STABLE SYSTEM UNDER PROPORTIONAL CONTROL

In proportional control, the control action is proportional to the deviation Q where Q is equal to the difference between the setpoint and the measured value. When the measure value is higher than the set value, the deviation Q will be positive and the correcting signal will be negative and vice versa.

Mathematically this can be written as

$$Y = -kp\ Q$$

The problem with this type of control is that it will give rise to output offset i.e. constant error between set value and the output value. One way to reduce offset is to reduce throttling range. Reducing the throttling range too far will lead to instability. The more quickly the sensor feels the control response, the larger the throttling range has to be to produce stable control. Also the value of 'should be' kp chosen in such a way that it makes the system response faster as well as it should not affect the stability of the system.

Facts about Proportional Control

1. The proportioning action occurs within a "proportional band" around the setpoint. Outside this band, the controller functions as an on-off unit, with the output either fully on (below the band) or fully off (above the band). At the setpoint (the midpoint of the proportional band), the output "on: off" ratio is 1:1; that is, the 'on-time' and 'off-time' are equal.

2. In proportional control, a unique value of the measured variable corresponds to full travel of the controlled device and a unique value corresponds to zero travel on the controlled device. The change in the measured variable that causes the controlled device to move from fully closed to fully open is called the throttling range.

3. The type of action dictates the slope of the control response. In direct-acting proportional control response, the output will rise with an increase in the measured variable. In a reverse-acting response, the output will decrease as the measured variable increases.

4. Summarizing….

    - Setpoint is desired value
    - Control point is actual value
    - Error or Offset = Setpoint - control point

PROPORTIONAL PLUS INTEGRAL MODE (PI)

Many control applications require a controller that can eliminate offset due to load and can control very close to setpoint. The controller always tries to match the setpoint and every time the load changes, the controller attempts to make the setpoint and the control point the same. PI control measures offset or error over time. The error is integrated, and a final adjustment is made to the output signal from a proportional part of this model. PI control response will work the control loop to reduce the offset to zero.

Integral mode adds a gain component algebraically to the controller output. This component is time proportional to the difference between the setpoint and the stable control point produced by the controller's proportional gain. This difference is caused by the offset due to load and is called steady-state error, which is the error between control point and setpoint when a balance between the load on the system versus the system capacity output and controller output is established. Steady-state error differs from the transitory error between setpoint and control point due to an upset in the process, such as a changing load or a step change in setpoint.

The integral mode adds a component of output to the output of the controller that is produced by the controller's proportional gain. The size of the component is determined by the integral gain multiplied by the error. As the error decreases, the size of the component integrated to the output signal also decreases, and becomes zero when the controller is controlling at the setpoint. This added component of output causes the valve actuator stroke position to change. The change in valve capacity resulting from the change in the actuator stroke position upsets equilibrium or prevents equilibrium from occurring until the control point reaches setpoint. This action causes two things to happen. The resulting temperature change due to the change in valve actuator stroke position causes the proportional gain to change its component of output signal at a magnitude proportional to the change in the temperature being sensed. While this is happening, the error changes because the control point is closer to setpoint, and this in turn causes the integral gain to make a change in the magnitude of its component of the output signal due to the changing value of the magnitude of the error between control point and setpoint. There will be no further change in the proportional-mode or integral-mode output components when the steady-state error is zero. Because of the combined action of both of these control modes, the controller can reduce the offset to zero, or nearly zero, and can establish a steady-state equilibrium of HVAC system control at a value very near setpoint. This type of control is called Proportional-Integral (PI) control.

Example: This approach would be applied, for example, to space temperature control in circumstances where the load fluctuated widely over relatively short periods of time. This could not be achieved by proportional control alone since the proportional band would have to be too wide. Also PI control is used more generally for applications where close control is required.

Other examples of PI control in buildings include mixed-air control, duct static pressure control, and coil controls. Assume that the room that is being controlled has an oversized valve. While there is no substitute for a properly sized valve, Integral control can make this condition less objectionable. To stop the actuator/valve assembly from hunting, a wide throttling range 8°F (4°C) may be set.

However, room temperature swings of 8°F (4°C) are typically unacceptable and complaints may become commonplace. A way to correct this situation may be to use proportional plus integral, or PI control.

Advantages & Limitations

The main advantage of this type of control is that the offset can be reduced.

A well set-up PI control loop will operate in a narrow band close to the setpoint and not over the entire throttling range.

PI control loops do not perform well when setpoints are dynamic, sudden load changes occur, or the throttling range is small.

DERIVATIVE CONTROL ACTION

Derivative control involves a further development of integral action such that the controller output is a function of the rate of change of the controlled variable. This form of control, like integral mode, would not normally be used alone, but in combination with others.

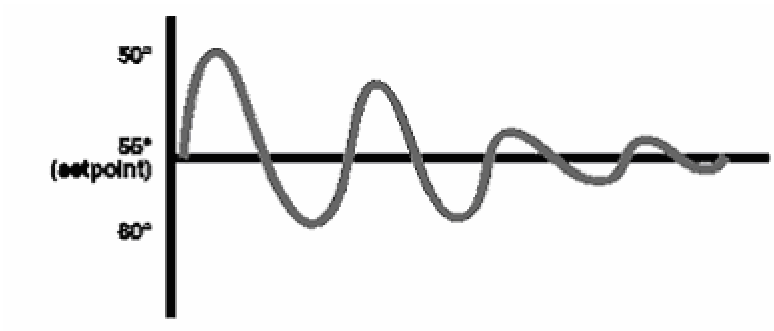
Effects of rapid load changes

The rate of change of load imposed on the HVAC system by the process affects how well the controller will perform its task of controlling at setpoint. The temperature of the outside air changes relatively slowly, and the temperature conditions inside also require some time to change. Inside conditions change as a function of air temperature changes made by the HVAC system, which warm up or cool down masses of material within the building. Inside conditions also change as functions of lighting load and occupancy. Because of these relatively slow rates of change, most of the HVAC processes that require gradual controller output changes can be controlled quite well with proportional plus integral (PI) control modes. Except for lighting loads on the HVAC system, these variable changes are relatively slow compared to the rates of change of variables that affect some non-HVAC processes. Lighting loads are sometimes imposed on the HVAC system quickly. This is an example of a step change in the process variable. The combined actions of proportional and integral modes are not always adequate to control rapidly changing variables.

PROPORTIONAL –INTEGRAL-DERIVATIVE (PID) CONTROL MODE

Some processes require a controller that can respond to rapidly changing process variables. One answer to control of rapidly changing processes has been the addition of another control mode called derivative mode. When this control mode is added to proportional-integral control, the combination is known as proportional-integral-derivative (PID) control mode. The PID control mode adds a component algebraically to the output signal; this component is proportional to the rate of change of the error between the control point and setpoint. This automatic adjustment also affects the proportional and integral output components in a manner analogous to the way in which the integral component affects the proportional component. As the valve actuator stroke position changes, the temperature changes as a result of changing flow through the valve, and the rate of error signal change between the control point and setpoint varies as the control point comes closer to setpoint.

PID is a precision process control application and is recommended in systems where the load changes often and the controller is expected to compensate automatically due to frequent changes in setpoint, the amount of energy available, or the mass to be controlled.



Proportional-Integral-Derivative Control

PID control locks the control system to the setpoint, narrowing the HVAC system operating range to just a few tenths of a degree, eliminating the widely varying temperature swings experienced in proportional control, and maintaining the zone temperature within tenths of a degree of the setpoint. As a result, the system uses the minimum amount of mechanical cooling or heating to maintain zone temperature.

There are a few HVAC control applications that are difficult for either P-mode or PI-mode control because of the fast rates of change of the process variables. One such application is the control of tankless heating converters, such as might be found in some domestic hot water heating applications. The ‘I-mode’ component of PI-mode takes care of the varying range of offsets due to loads that occur in domestic hot water heating applications.

For example, high-rise residential buildings have morning and evening peak periods of demand for hot water use. These peak demand periods drive the domestic hot water temperature to the low end of the offset range. Periods of relative non-occupancy, such as late morning and early afternoon, drive the temperature to the high end of the offset range due to the minimal demand for domestic hot water use. Periods of relative inactivity during occupancy, such as late evening and very early morning, require practically no domestic hot water. It is this period that defines the top end of the offset temperature range. The P-mode control alone does not control the water temperature very close to the controller setpoint in this kind of application. The addition of the 'I mode' to the 'P mode' makes the offset range much narrower than would occur with P mode alone. However, PI modes alone cannot handle the unpredictable diversity of demand as the peak periods start and end. What happens during the period of light demand for hot water use is that the turn-on of a shower or the startup of a dishwasher produces an upset in equilibrium that has a greater effect than the same event would have during a period of heavy demand. Periods of heavy demand tend to filter out some of the effect of a single turn-on.

In the control loop applications where manual tuning is prescribed, the proportional mode constant is set as the result of a calculation. These applications cannot use the integral or derivative control modes. Self tuning is prescribed for these applications because finding the optimum setting manually is difficult and time consuming. When controllers self-tune, these settings are automatically optimized, and an optimized derivative-mode setting is selected due to the controller's self-tune feature.

In general, PID is not much suitable for HVAC applications firstly because the most HVAC processes do not require rapid control response and secondly the operation is not very energy efficient. Application rules dictate that for effective PID operation, valves and dampers must be undersized so that a substantial pressure drop occurs across them under all operating conditions. When applied to chilled- or heating water distribution systems, this recommended valve pressure drop can as much as double the pumping power required. This loss is magnified because PID control moves variable-speed pumps and fans away from their "natural curves" (the curve of the pump's or fan's highest efficiency at various speeds), thus reducing operating efficiency while adding substantially higher operating pressure requirements.

STANDARD HVAC SYSTEM CONTROL LOOPS

The standard HVAC system control loops use the single-loop digital controller and additional components. These components are collectively called the control loop logic. The logic varies with the loop requirements, and its purposes are to interface the loops with the operational mode signals, to modify signals, and to interface with energy management control systems (EMCS). The control loop logic is implemented by the use of combinations of relays and function modules.

Relay coils are activated by occupied-unoccupied, ventilation-delay, safety shutdown, EMCS override, and other signals external to the HVAC control system. The contacts of the relays interrupt analog signals of controllers and function modules, provide inputs to on/off control loops (such as starter circuits), and operate HVAC control panel pilot lights.

All the relays, contacts, and function modules are defined for each loop. The relative physical locations of the relays and function modules will be assigned in the HVAC control panels.

Each control loop on the drawings shall be represented to show the elements of the control loop, such as sensor/transmitters, controller, function modules, relay contacts, current to pneumatic signal converter (IP), and final actuator.

Stability of the System

Stability of a control system is concerned with its response to a disturbance. The system must return to a steady state condition to be considered stable. An ‘Unstable’ system is characterized by an oscillating response, or an ever increasing (or decreasing) response until some natural limit is achieved.

An unstable system will often display a hunting behavior – the control valve is constantly varying (In an HVAC system, the controlled device should not change more than 20% in a 10 minute period).

This produces excessive wear on the valves and actuators, produces thermal cycling that may impact other equipment, structure, and contents of the building, or produce instabilities in other parts of the system.

Factors contributing to an unstable system:

1. Too much gain (too narrow a throttling range) for a proportional system.

2. The controlled variable has too much capacity to be reasonably controlled.

3. Lack of sufficient feedback to allow control (an open loop system).

4. Control mode is too simple (ON/OFF for example).

5. Incorrect install – Often, the sensor providing feedback is located in a remote location. Several, tens, hundreds may be wired back to an I/O panel and several sensors may be miss wired.

6. Too much lag time (delay) in the response of the system.

Oversizing of control devices would result in systems in which it would be difficult or impossible to obtain satisfactory control loop operation, regardless of the quality of the controller and components used.

The designer must not assume that self-tuning controllers will compensate for oversizing of control devices.

The "I and D" modes compensate for HVAC system load variations and HVAC system equilibrium upsets, but do not compensate for incorrect valve and damper sizing.

ENERGY MANAGEMENT OPPERTUNITIES

From the energy management perspective, the most important function is to optimize control. The concept of optimizing control is not only to control space conditions, but also to do it in a manner that minimizes the energy and costs when different forms of energy are available. An optimizing strategy is generally to improve the efficiency of primary supply equipment or to reduce the losses of energy in end-use systems.

Here are some of the critical control strategies that can save energy.

1. **Select realistic operating hours -** For any building that shuts down overnight, every extra hour per day of operation represents approximately 7% additional air-conditioning energy. Don't run the building for a handful of early birds or night owls - generally conditions in the building will be near enough to normal comfort levels without extended hours.

2. **Select realistic space conditions -** Controls should be set up to provide a dead band between 68°Fand 73°F where neither heating nor cooling will occur. This band reduces starting demands on plant and saves energy by recognizing that people dress differently during heating and cooling seasons.

3. **Logical zoning of HVAC areas -** Zoning in a manner that areas of similar load characteristics are controlled together can reduce energy use considerably by avoiding the amount of re-heat needed to maintain conditions. In general, this is only possible at the time of construction or major refurbishment. However, changing building use may mean that sensors can be relocated to better reflect building conditions. In some cases, relocation of partition can isolate sensors from the systems they control, under which circumstances it is essential to relocate the sensor back into the relevant conditioned space.

4. **Early morning warm-up or cool-down -** The strategy is to seal the building from the introduction of outside air and apply maximum heating / cooling to achieve design conditions in the minimum time. It is necessary to establish that the building will not be occupied during this period and that the building has enough thermal storage to warrant using the system.

5. **Night time cool flush -** This strategy uses "low heat" night air to cool the structure of the building when the internal temperature is above the lowest comfort temperature. An additional advantage is that internal air quality will be improved. When evaluating the economics of this system, assess the power requirements of the fans required. It may be no cheaper than using the chiller plant for a short period before occupancy; however this technique does not improve air quality.

6. **Fresh air control -** When it is cooler outside than inside, it is often possible to use outside air to provide cooling.

7. **Scheduling -** Sophisticated control systems with 365 day clocks can be scheduled to ensure that plant does not run on all days offices are not in use. Starting and stopping times can also be changed when daylight saving commences and ceases. Continued management is required to update the program for changes such as moveable feasts and so on.

8. **Optimum start/stop routines -** These routines monitor the time taken for the building to reach design conditions in the morning and to depart from design conditions when the HVAC is shut off at night. The start and stop times are progressively modified over a number of days until a good match with the building requirements is achieved. The optimization routine continues to modify the start and stop times as the seasons change.

9. **Improving Part Load Performance -** The sizing of equipment is to meet maximum loads, but the equipment is usually run at less than maximum load. This means that the part -load characteristics of the equipment determines the efficiency in meeting a given load. When there are multiple chillers or boilers, an optimizing strategy would be to choose the most efficient equipment that has the capacity to meet the part load at any given time.

While the modern day controls, have immense potential to capture data; it is useless unless it can be associated with other date to provide some useful information. Don't try to load your control system too much. Remember, when it comes to control, it may be better to have a building that is dumb and reliable than a building that is smart but scatterbrained. A simple strategy to minimize risk in this respect is to keep the controls simple and robust rather than complex.

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